BUREAU OF RECLAMATION HYDRAULIC LABORATORY

RIPRAP STABILITY IN CHANNELS

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PAP

by

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Riprap is used in channels to dissipate the excessive energy of flow which would otherwise erode the channel and thereby decrease flow efficiency. Flow efficiency or discharge capacity is apt to be reduced at points of scour by such forms of discontinuity as vorticies and boils which introduce air into the water as well as impede its downstream movement. Several theories have been proposed in an attempt to arrive at meaningful design criteria for the size of riprap needed in a given situation. Because the problem is so complex, reliable experimental data concerning riprap is scarce.

In this study, the writer attempts to analyse the problem on the basis of forces resulting from boundary velocity distribution at the riprap surface. Velocity profiles were measured and recorded with the aid of a differntial pressure cell for the following two conditions:

- Initial instability-defined as the removal of several pieces of riprap from their original location leaving a scour hole or cavity.
- Scour to a depth equal to the largest size of riprap present in the test category.

The information obtained by this research will lay the groundwork for future studies in riprop gradation. This provides an interesting comparison with formulations published to date for specifying riprop sizes in prototype design. The tests discussed only concern size categories which fall near the size versus velocity curves shown at the end of the report. However, it must be noted that the curves

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published by the U.S. Bureau of Reclamation and others represent the 50 per cent passing material in a well graded riprap sample. The pressure distribution immediately above the rock surface is investigated to check possible uplift tendencies. Static pressure wall ports and a multiple Prandtl tube probe were used.

Shape as a parameter was investigated by comparing data taken from both cobbles and angular material. Careful measurements of physical characteristics were made on a random sample from each size category of test material. Measurements included volume, weight, and major axial dimensions. Photographs were also taken before and after one erosion test for each size category. An attempt was made to compute the scale factors of material measured as a preliminary step for proposed model tests of full scale riprap gradations.

This abstract is approved as to form and content.

Signed

Faculty member in charge of dissertation

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NOMENCLATURE

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A_c = Gross-sectional area of the contracted jet of the flow, ft.² a, b, c = axial dimensions: longest, intermediate, and shortest, respectively, inches.

c, c1, c2 = Constants of variation.

- c_{2} = Coefficient of energy loss which is a function of the contraction of the subserged flow jet under the baffle, dimensionless.
- cv = Coefficient of energy loss which is a function of velocity of flow, dimensionless.
- D = depth of the velocity jet under the baffle (modified from baffle opening for jet contraction), ft.
- d = Characteristic linear dimension for sediment size, inches.

d_s = Diameter of equivalent weight sphere, inches.

d_c = Edge of equivalent weight cube, inches.

- d_{sc} = A linear size measurement which combines characteristics of both cube edge and sphere diameter (see eq. 1.4, p. 6).
- f = Frequency of repetition, cycles per second, of similar velocity
 fluctuations.

g = Acceleration of gravity.

 $h_A =$ Amplitude of the velocity head fluctuation, fps.

 $h_{\rm R}$ = Energy loss at the baffle, ft.

- K = Special coefficient of variation developed by Hedar (see p. 10).
- L = Characteristic eddie length as defined by Dryden, ft. (see p. 26).

n = Porosity, uimensionless.

p₀ = Static pressure, psi.

 $p_s = Stagnation pressure, psi.$

q = Discharge per foot of channel width, cfs/ft.

- $q_i =$ Jet discharge per foot of width under the baffle, cfs/ft.
- R(y) = Correlation coefficient of longitudinal fluctuation of velocity
 - (component normal to the flow tube section, see p. 26).
- s = Shape factor, dimensionless.

sp. gr. = Specific gravity, dimensionless.

- u = Total longitudinal velocity component at a point, fps.
- **ū** = Mean longitudinal velocity component at a point, fps.
- u¹ = u ū = Turbulence fluctuations about the mean velocity at a point in the flow, fps.
- V = Volume of riprap material, ft³.
- ∇ = Mean point velocity, fps.

v = Mean velocity in the channel of flow, fps.

v_b, v_m = Mean water velocity for incipient or beginning motion of a given size aggregate, fps or meters/sec.

y = Effective depth of flow, ft.

 $y_v = V$ ertical distance from a median point in the rock surface profile to a given point in flow velocity, inches.

W = Weight of individual rock, 1bs.

- W_{sc} = Equivalent weight combining characteristics of both cube and sphere (see eq. 1.3, p. 6), lbs.
- Slope of channel at the centerline in the direction of flow, dimensionless.
- X_s = Unit weight of stone, lbs/ft.3

 $\delta_{\rm W} =$ Unit weight of water, lbs/ft³.

 Θ = Position function, dimensionless.

 μ = Dynamic viscosity of water, lb-sec/ft².

 $\pi = 3.14159$

 ρ = Term used for natural slope of randomly placed rubble (similar to angle of repose, see eq. 1.12, p. 11).

 ρ = Density of stone, lb-sec²/ft⁴(see eq. 1.13, p. 13).

 $P_s = Density of stone, lb-sec^2/ft^4$.

 $f_{\rm W}$ = Density of water, 1b-sec²/it.

 $\sigma =$ Density of water (see eq. 1.13, p. 13), lb-sec²/ft⁴.

 $\not =$ Natural angle of repose of material measured from the horizontal. $\tan \not =$ Coefficient of internal friction.

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CHAPTER 1

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BACKGROUND AND THEORETICAL CONSIDERATIONS

Riprap protection is an important feature of water conveying systems. Since early times, man has known that he could protect his property from erosion by placing rocks and other forms of rubble in the area of scour. As the population density along waterways became heavier, the need for more sophisticated methods of providing erosion protection was made evident.

It is the objective of the writer to approach the problem in a pragmatic manner to ascertain what the problems involve, and how the modern researcher can pursue the following questions:

a. What are the factors of stability?

b. What stability factors are significant to the designer?

Techniques for choice of riprap size and specifications have lacked the basic information which could be found in the laboratory. This deficit of information is partially due to the difficult problem of isolating any of the many parameters responsible for the erosion of riprap. The equipment used in this study establishes boundary velocity conditions by using a sluice gate type of movable baffle to induce a submerged jet along the riprap surface. This study investigates only the conditions present on the stream bed and does not consider embankment problems.

Riprap Stability

The immediate question arises; what are the stability requirements of riprap? Riprap has been defined as..."a layer, facing or protective mound of stones randomly placed to prevent erosion, scour or sloughing of a structure or embankment; also the stone so used (1)*." The Bureau of Public Roads broadened this definition to include mortared and grouted riprap, concrete in bags, concrete slab riprap and stone riprap for foundation protection(2). These descriptions point to the importance of weight and flexibility for stability. Closely related to weight is the effective size, gradation, shape and general proportions of the overall blanket. Finally, the manner of placement may be the sole reason for instability of the blanket, and a major factor in the cost as well.

Along with the physical characteristics of the blanket, the behavior of the riprap when exposed to hydrodynamic forces must be analysed. Turbulence resulting from various forms of roughness, velocity patterns, and the resulting pressure fields in the fluid all result in significant forces which can dislodge portions of the protective blanket.

Cost is a dominant consideration in every aspect of riprap construction. Consequently, the handling of the heavier pieces becomes an expensive factor in placement. Conflicting with this desire for less weight however, is the fact that none of the other parameters seem to be a sufficient substitute for weight as a stablizing factor. Riprap blankets have failed when the thickness was doubled to compensate for the use of lighter materials. Another

* Numbers found in brackets, as (1), refer to the bibliography on page 39.

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substitute for weight has been the use of various techniques to increase the size of the unit, for example: tieing several sections together with wire screen, cable and mortar to link individual or groups of stones or manmade materials into one flexible unit (2) (3) (4). Naib (5) and Carey (4) give information about other forms of manmade riprap which have great promise for the future. Also very important is the type of foundation constructed for the riprap. The type of soil filter used can be instrumental in determining whether or not the bank will erode between the larger portions of riprap (2) (6) (7). Length of the riprap blanket is determined by estimates of the area affected by eroding forces such as high velocity jets, turbulent eddies and vortices (2). Side slope, centerline grade and hydraulic radius all play an important part in this analysis. Better methods of predicting future sources of change in the watercourse and their effect on stability requirements would be of considerable help to the riprap designer. For the purposes of this report, however, the discussion will be restricted to some of the more fundamental stability parameters of riprap. These parameters shall include rock size (weight and volume), shape (round versus angular), and water velocity and pressure variations with depth and time.

Riprap Failure Sequence

An analysis of the sequence of events occurring at the time of riprap failure should shed some light on the areas for study. From general observations in the laboratory and natural stream beds, it has been noted that as the velocity at the riprap surface is increased, there is a slight rearranging of the material according to shape. A slight quivering becomes increasingly evident among the smaller rocks.

The quivering seems to be directly related to the cross-sectional area of the individual rock exposed to the current and its position with relation to adjacent segments. Failure is often initiated by the removal of a key stone behind which a line of other stones has been formed by current patterns. A whole line at a time may fail and wash downstream a significant distance. A portion of the blanket is then cratered, and its resistance to future erosion reduced. The larger the stone size, the less warning occurs before failure. The stone may be lifted suddenly high into the current of flow, as in Figure hB. To the contrary, the larger material, if it is shaped right, may merely adjust slightly to a more stable position and remain stationary.

PARAMETERS OF STABLITY

From the above sequence, it can be seen that certain difficultto-define non-uniform parameters exist in the riprap installation that make analysis difficult. The rock shape and position may vary considerably. Velocity currents can form in a myriad of different patterns and time sequences. Fluctuations in flow cause multiple variations of pressure forces on the riprap.

Shape

Shape can be described as stream rounded cobbles, hereafter referred to as "cobbles", and quarry-blasted, crushed "angular" material. Shape can also be significantly categorized by dimension ratios separating rods, disks and blades from spheres (7) (8) (9) (10). Ratio limits should be determined for maximum variations allowable in length to width thickness. The Army Corps of Engineers makes

the following comments about shape: "The shape of rock that tends to break due to quarry operations or other reasons into elongated or slabby particles needs to be controlled by provisions in the specifications. It is desirable not to have stone whose long dimension materially exceeds their short dimension." Cubical stones "nest" together best and are most resistant to movement. Limitations should be specified as follows:

- a. Stones having the ratio of longest to shortest dimensions greater than two should not exceed twenty-five per cent of the total.
- b. Stones having the ratio of longest to shortest dimensions greater than three should not exceed five per cent of the total riprap material. (11)

Gradation

Gradation follows directly as the next point of interest. There is the question of which portion of the gradation should make up the largest portion of the blanket-the larger sizes or the fifty per cent passing or some other portion of the blanket? Should this be determined by gross volume or weight? What is the most desirable per cent of smalls to be included? The Bureau of Public Roads, the U.S. Army Corps of Engineers, and the Bureau of Reclamation have all written specifications for riprap.

Rock Size

The specification of the size of individual pieces of riprap is further complicated by the different methods used. One method that seems to be prevalent specifies size by the diameter of sphere of volume and weight equal to that of the individual stone (11). Reduced to equation form:

(1.1) Weight =
$$M = (V_s \pi d_s^3)/6$$

where $X = unit weight of stone. lb/ft3$

 d_s = diameter of equivalent weight sphere.

(1.2)
$$d_s = (GW/\delta_s \pi)^{\frac{1}{3}}$$

 $d_s = (W/0.52 \delta_s)^{\frac{1}{3}}$

A second form combines the characteristics of both cube and sphere as follows (11):

(1.3) Weight = $W_{sc} = \frac{1}{2} \left[d_c^3 \delta_s + \delta_s (\pi d_s^3 / 6) \right],$

where d_c = edge of equivalent weight cube.

Calling the new dimension d_{sc}, for cube and sphere dimensions combined, and solving for it:

(1.4) $d_{sc} = [2W/(1 + \pi/6)\delta_s]^{\frac{1}{3}} = (W/0.76\delta_s)^{\frac{1}{3}}$ Conversion from one to the other form may be made by applying the appropriate factor:

$$(1.5)$$
 $d_s = 1.13 d_{sc}$.

For studying the various size to velocity curves, both forms of d will be used (d_s for cobbles and d_{sc} for angulars) since the curve shifts according to the size parameter used.

Blanket Thickness

Blanket thickness is directly related to the particle size. Some designers (11) specify that the thickness of the riprap layer shall be equal to or greater than one hundred per cent passing or 1.5 times fifty per cent passing, whichever results in the greater thickness. The object is to be sure that all stones are fully contained within the riprap layer.

Velocity as Index to Current Erosion Potential

Riprap installations are surrounded by varying pressure and velocity fields resulting from turbulent flow in and around the rocks that make up the blanket of protection. Of main interest to those concerned with riprap throughout its history has been the so-called competent velocity at which riprap erodes. Much terminology and data have been drawn from the closely related subject of sediment transport. Competent condition is the term used by Berry (12) to describe the limiting case between the vibration of bed-load and the start of uninterrupted movement of the pebble.

The cause or indicators of the forces present in a given situation have been separated into two general categories. These are velocity scour and turbulent boil or scour. Both forms relate to flow characteristics in the channel.

Velocity versus Riprap Size

The use of velocity as an indicator of riprap size requirements in a channel has been widely practiced. The number of particular velocities considered significant are almost as numerous as the originators. The mean velocity was generally considered a good index until a number of researchers presented velocity-versus-size curves that showed how sensitive the upper range of sizes were to even a slight change in velocity. From these curves, (Figures 3h and 35) it can be seen how a slight error in the velocity chosen as critical could radically change the stability capability of the riprap layer.

Berry (12) defines a competent mean velocity, sometimes known as the critical mean velocity, as the velocity at which a certain

size sediment on a channel bottom is just moved. He states that there is a limitation to mean-velocity criteria in that it does not bear a constant relationship to the bottom velocity. It is the bottom velocity which is the probable controlling factor in bedload transportation. Likewise bottom velocity is significant in riprap instability.

Another reference to the mean velocity was made by Etcheverry (13) in 1915. He states: "Bottom velocity is approximately 0.75 of the mean velocity." This figure was used by Peterka (14) in estimating velocities present in several prototype examples which both survived and failed under excessive flow conditions.

Berry (12) reports that ... "Bogardi, on the basis of his experi-

(a) The roughness factor denoting the bed friction varies independently from the critical mean velocity as well as from the critical bottom velocity. He therefore concluded, that from the point of view of bed friction alone, it is the same to characterize the competent condition by the mean velocity as by the bottom velocity.

(b) Within the limits of his experiments, for the same bed-load the competent mean and bottom velocities were not influenced by the width of the flume. From this viewpoint also, there is no difference whether we use the critical mean or competent bottom velocities for the criteria for start of movement.

(c) He further found that for the same bed-load material the competent bottom velocity does not depend upon the depth of water in the channel. On the contrary, however, the mean velocity varies

with the depth of water, increasing with increasing depth of water."

Berry (12) further reports that..."as a result of analysing Gilbert's experimental data, Rubey has shown that it is the velocity in the immediate vicinity of the particle on the stream bed that is significant for the start of bed-load movement. He defines the bed-velocity as the velocity at the boundary between the thin film of laminar flow on the stream bed and the main mass of turbulent water above it."

The "sixth-power law" represents a commonly used relation between the eroding power of the current and variation of the velocity. John Leslie (12), Wilfred Airy (15), and S. V. Isbash (16) have been credited for contributing to the development of this relationship. The sixth-power law states that the weight required for stability varies as the sixth power of the velocity acting on the rock. Isbash's curve interpreting the power law is found in Figures 35 and 36. Figure 35 also gives several arbitrary curves presented for use in specific situations. Campbell says, "To estimate the required size of rock which is stable for a given velocity, the fifty per cent passing is used." "A coefficient $E_s = 1.20$ is used as the basis of the Isbash curve." (17) (See Figure 35) Superimposed on the chart will be found data points resulting from the present research for comparison.

Berry (12) introduced the following relation:

(1.6) $V_{\rm b} = 0.51 \sqrt{d}$

where d represents sediment sizes from a quarter of an inch to twenty-six inches. Some data points from this curve may be found

on Figure 35. The bed-load material represented is of an average specific gravity of 2.65. Movement includes rolling or sliding on a bed of the same size, and for average turbulence and particle shapes.

Figure 36 summarizes the findings of Soren Andersson (18). For ease of comparison, notation is changed to coincide with that used throughout this report. The following is an excerpt from Andersson's report:

"SHIELD'S (1936)
(1.7)
$$d = \frac{l_{\omega}}{l_{z}-l_{\omega}} \left[\frac{y \sin \alpha}{0.06} \right]$$

where $\ll = \pounds$ slope of channel in the direction of flow. MEYER, PETER & MULLER (1948)

(1.8)
$$d = \left[\frac{V_{w}}{V_{s} - V_{w}}\right] \left[\frac{y \sin \alpha}{0.03 \text{ to } 0.047}\right],$$

where 0.03 must be assigned to complete rest and 0.047 to a situation in which material transport is zero. The last value was obtained by extrapolation of the test values in measurements of material transport.

SUNDBORG (1956)

(1.9)
$$d = \frac{Y_{\omega}}{\xi_s - \xi_{\omega}} \left[\frac{3 \ y}{2 \ c_1 c_2} \right] \left[\frac{\sin \alpha}{\tan \phi \cos \alpha - \sin \alpha} \right]$$

where c_1 and c_2 = coefficients dependent on degree of turbulence and state of density of the material (normally, $c_1 \cdot c_2 = c_1$

may be fixed at 0.087).

HEDAR (1960, 1962) (1.10) $d = \frac{\chi_{uq}}{\chi_s - \chi_{us}} \left[\frac{1}{|K|}\right] \left[\frac{\nu_m}{\log\left(\frac{14.63}{4}\right)}\right]^2 \left[\frac{1}{\tan \phi \cos \alpha - \sin \alpha}\right]$ where ν_m = mean velocity of water, m/sec

K = coefficient evaluated by Hedar = 7.5 . For the two equations above, ϕ = angle of natural slope of the stones.

ISBASH (1936)

(1.11)
$$d = \frac{l_{\omega}}{l_s - l_{\omega}} \left[\frac{\upsilon_m^2}{2g}\right] \left[\frac{l}{1.44}\right]$$

where g = acceleration of gravity, m/sec²." (18)

The coefficient 1.44 has been obtained empirically on the assumption that the blocks of stone are dumped pell-mell in running water. Andersson further states that his illustration of the above equations and others represent flow on a gentle slope at a depth of water equal to twenty rock diameters (Figure 36).

The California Highway Department, in a publication entitled <u>Bank and Shore Protection in California Highway Practice (3)</u>, gives specifications for choosing rock size where current velocity governs. Since theirs is for embankment rather than channel bottom protection, it is only mentioned here for general interest. Their nomograph (found in the Appendix) is based on the following equation:

(1.12)
$$W = \frac{2 (10)^{-5} \sqrt{3} \overline{g_r} (5 c^3 (p - \alpha))}{(\overline{s} \overline{g_r} - 1)^3}$$

where V = velocity of the stream near the bank to be protected \overline{Sg}_r = specific gravity of the stones

 \mathcal{P} = 70° for randomly placed rubble

~ = embankment face slope

W = minimum weight in pounds of outside stone for no damage.

It is interesting to note that the emphasis is on weight and velocity with a correction factor for specific gravity. They have related random-shaped boulders and broken stone to some of the commonest geometric shapes which form a shape spectrum extending from fully rounded to fully angular. Tabulations of factors for precisely

defined shapes will provide a basis for interpolation of approximate or intermediate variations (see table and illustration of shapes in Appendix).

The U.S. Bureau of Public Roads has published a report summarizing various uses of riprap for bank protection (2). Their stone-weight versus velocity curves were taken from the same original source as that used by Campbell (19) published and supplemented from time to time by the U.S. Army Corps of Engineers at the Waterways Experiment Station in Vicksburg, Mississippi. For better comparability, the supplementary curves added by Campbell were superimposed on the figure used by the U.S. Bureau of Public Roads along with data points from Berry (12), Peterka (14), and the present research.

M. G. Hiranandani, director of the Central Water and Power Research Station at Poona, India, has written a review and comparison of his findings during visits to hydraulic research laboratories in the United States, United Kingdom and France with his activities at the Poona Research Station (20). He tells of experiments that were conducted at the CWPRS, Poona, in 1940, for determining the size of stone which should be used to represent the stone of block protection given in prototype structures at bridges or else at weirs and falls. "Tests with stones were made both for angular stones and rounded stones. The velocities required for initial continuous movement, and also for general movement were observed. The stones were laid over a smooth bed and alternatively on fixed layers of stones of the same size. The data collected during these experiments for angular stones and for initial movement enabled the following correlation to be established." (20)

(1.13)

$$\frac{\text{Dia.}}{\text{d}} = 0.1145 \left[\frac{\text{V}^2}{\text{gd}} \cdot \frac{\mathcal{F}}{\sigma - \mathcal{F}} \right]$$
(1.14)
Dia. = $0.1145 \left[\frac{\text{V}^2}{\text{g}} \cdot \frac{\text{V}_5}{\mathcal{E}_{w} \cdot \text{V}_5} \right]$
where d = depth of flow in feet
Dia. = size of stone-diameter in feet
 $v = \text{average velocity in feet per second}$
 $\mathcal{F} = \text{density of the stone}$
 $\sigma = \text{density of the water.}$

It should be observed here that the transporting power of water is different from its erosive power and the two should not be confused. As Berry (12) describes it, transporting power must overcome the weight, and erosive power must break the cohesive bonding of the material. "The latter varies as the square of the bottom velocity while the former varies as the sixth power of the velocity." It has been observed that in many cases of removal of slightly cohering material, the resistance is a mixture of both, and the power of removing material will vary as some rate between v^2 and v^6 . It is the cohesive force that causes a major difference between prototype and laboratory results. Impurities found only in the prototype waters form various kinds of bonding on the riprap material affecting

the erosion patterns of the smaller crushed material and, in particular, the filter blanket underneath.

Discharge may be a pertinent parameter for ribrap instability. Joonejo (21) concluded from his study of rock movement under flowing water that..."for a given slope and sample, the loss of material increases as the discharge increases." This problem of varying discharge is investigated in the present study.

Dimensionless parameters are often used to help describe significant characteristics of hydraulic systems. These parameters are arrived at by dimensional analysis of a variety of combinations of geometric, kinematic and dynamic dimensions. Reynolds and Froude numbers are familiar dimensionless ratios often used. Vennard (23) and Streeter (2h) both give excellent developments of dimensional analysis including the Buckingham II-theorem.

Buckingham showed that, if n variables were functions of each other, k equations of their exponents could be written. Dimensional analysis would assemble the variable into (n-k) dimensionless groups which are functionally related. Buckingham designated these dimensionless groups by the capital Greek letter Pi (II). The II-theorem offers an advance notice of how many groups are to be expected and some clue as to their formulation.

Char Tool 2

LABORATORY QUIPMENT AND PROCEDURE

Tests were made in the large flume at the U.S. Bureau of Reclamation's Hydraulic Laboratory in Denver, Colorado. As can be seen in Figure 2, only a portion of the flume was used for the actual test section in order to cut down on the amount of rock that would have to be handled. Of the overall flume which is four feet wide by eight feet deep by eighty feet long, the test section containing gravel was reduced to two feet of width and twenty-eight feet of length. The test scour section extended downstream from the baffle approximately ten feet, but only a portion of this section was usually affected by the submerged jet. One side of the flume is glass-walled thus lending itself very well to sectional erosion tests. The baffle carriage and adjacent equipment (Figures 1 and 2) were already assembled for use in previous tests concerning the cleaning of manmade salmon spawning beds in canals (22).

The fundamental function of the equipment used for this study was to provide a relatively homogenous velocity jet in the region near the boundary of the rock surface. In this way, it was hoped that the velocity most significant for the scour of the particular rock would become more evident. By reducing the number of variables, the research engineer can better examine a complex problem. The passage of the water under the baffle makes negligible the effect of the upstream velocity distribution on the test section scour. Turbulence, though certainly present, is of a much more uniform pattern than found in the field. Turbulent surging often appears to be the major cause for severe scour in prototype structures.

Analysis of the surge pattern requires a model of the specific situation in each different individual structure, and thus cannot be dealt with in a general study of this type. Figure 1 shows a schematic representation of how the velocity of the submerged jet was increased by lowering the baffle and thereby increasing the differential head which drives the jet.

Several discharges were used to determine its effect on riprap instability. The Bureau Laboratory uses a typical recirculating system where the water is driven by centrifugal pumps through a pipe system to the model and finally dumped back into a large capacity sump. Discharge was measured by a standard Venturi Meter connected to a mercury differential manometer. Twenty-five cubic feet per second was the maximum flow that could be produced by the system.

Velocity distribution was recorded in the vertical plane parallel to and passing through the centerline of the two foot cross section. The one foot scale was included in the pictures showing the scour patterns (Figures 5 through 9) to indicate the channel centerline. The vertical velocity profile was measured first with one Frandtl type pitot-static tube and later with two such tubes nounted in tandom, one above the other. In the latter case, the two tubes were spaced a vertical distance apart equal to the largest size of material in the category being tested. For the six to nine inch angular material, the tube spacing was reduced to give more velocity readings for the velocity jet.

The horizontal velocity distribution was assumed to be symmetrical about the centerline. Since the flume sides were

relatively smooth compared to the rock surface in the bottom and considering the magnitude of the velocity in the jet, the symmetry assumption was considered adequate.

Velocity is measured by recording the differential pressure represented by the total (stagnation) pressure minus the static pressure: $\tau = \sqrt{2g [p_s - p_o]/8}$ (2.1)

where ps = stagnation pressure

p = static pressure (23).

For this study differential pressure transducers of one and five pounds per square inch capacity were used to measure the velocity head. Several brands of tape recorders were used to produce a graphical record of the pressure fluctation with time. Part of the tests were recorded through an averaging circuit, but for the most part, the pressure differential noted above or velocity head was averaged by eye. The resulting data compared favorably with that obtained with the averaging circuit. Even the mean value recorded using the averaging circuit did not remain constant. The recorders were recalibrated before each test velocity profile and at the conclusion of the test to check for zero drift and span settings. Recording inaccuracies, when noted, were prorated between calibrations with added weight given readings with a high degree of fluctuation. Experience has shown that large, frequent pen movements tend to shift the zero setting slightly and sometimes change the span setting. A sample of the recordings may be found in Figures 10, 11, and 12.

Reduction of velocity was done by estimating the area on either side of a straight hairline etched on plastic which was placed over the recording trace for a given point. The line was adjusted by

eye until approximately equal areas were bound by the trace and the hairline. If drift was noted so that the hairline was not parallel to the grid, the points at which the beginning and ending of the trace intersected the hairline for a particular measurement were noted, and average reading taken at the mid-point on the hairline. In this way, the average velocity at a point for several seconds was taken. Velocity profiles were drawn (Figures 17 through 29) showing the vertical profile of the velocity measured relative to an arbitrarily defined datum. The datum was defined as a maximum envelope level line at the extremities of the riprap surface for tests 1 through 12. For the remainder of the tests, the datum was taken as a preset level line that approximated the average rock surface. That is, some rocks projected above the line and some fell below.

The two states of stability shall be defined as follows:

1. Initial instability shall be that condition when several rocks move from their initial positions indicating a condition of unbalanced forces.

2. Maximum scour shall be restricted to one maximum screen size dimension of the category being tested.

Test Procedure

1. Rocks were sized in categories according to specified screen sizes. The cobbles and angular material were tested separately.

2. Random samples were picked from each category, identified by number and sized as discussed above. The volume was measured by water displacement. The physical features a, b, c, and V may be found in Tables 1 through 12.

3. Rocks were placed in the flume at a prescribed depth according to the datum used, and the plan view was photographed (Figures 5 through 9).

4. Flow was established with the baffle wide open.

5. The velocity of the submerged jet was increased slowly for constant discharge by lowering the baffle. Time was allowed for the backwater time lag of the system so that the effect of each incremental gate closure could be identified. Otherwise the critical value of velocity for instability might be passed by.

6. The baffle opening was recorded for the condition of initial instability as discussed previously.

 The opening was also measured for complete scour to a depth equal to the larger screen size of the category being tested.

8. The velocity profile was then taken.

9. The water depths both upstream and downstream were taken from time to time during the velocity measurements.

10. Discharge readings were also made periodically to check against any drift resulting from changing tailwater conditions.

11. Pictures were then made after the maximum scour to show the erosion pattern of the numbered sample rocks.

12. Profiles were drawn of the cross section under the baffle and along the channel centerline to indicate the depth of scour and to estimate the effective flow area under the baffle.

Separate tests were run using several different discharges and depths for several material categories to establish the effect of discharge and blanket thickness as variables to be related to scour velocities. Velocity tests were run twice for most of the rock categories. Rock blanket thickness, discharge and rock placement were kept as nearly identical as possible.

CHAPTER 3

RESULTS AND CONCLUSIONS

Data Analysis

Riprap shape determines how the separate pieces will fit together and equally important, how stable it is when exposed to hydraulic forces. The importance of shape has been observed, but practical methods of its determination have been lacking. Several methods are recommended in this report as being helpful in relating a particular riprap sample shape to a simple sphere or rectangular solid.

Description of riprap in place in the prototype is nostly in the form of pictures. A few measurements of size have been made in the field. Ball, Lancaster, and Schuster (25) of the U.S. Bureau of Reclamation discuss a technique used at Grand Coulee Dam in the Columbia Basin Project. "Groups of rocks at various stations on both sides of the tailrace banks were located by survey and marked before high water of May, 1949. Individual rocks of the groups varied in shape and weight and were chosen at random to represent a general gradation of riprap in the area. After the flood season, a re-survey noted the movement of the rocks that remained. Many of the rocks and some groups were not recovered because of readjustments of the rioran cover. Those rocks moved and recovered after the flood indicate the extent of readjustment (25)." In a similar manner, rocks from each category in this study tested were numbered, weighed, and the volume and three axial dimensions measured. The numbered rocks were picked at random from the surface of the test section. After they were measured, they were returned to the top layer with

the expectation that they would be the first rocks to become unstable and scour. Their movement during scour was traced with pictures taken both before and after scour (Figures 5 through 9).

Riprap shape can be described several ways. Zingg (7) developed a classification of particle shapes based on the four general shapes shown below.

Class	Shape	b/a	c/b
I	Disks	> 2/3	< 2/3
II	Spherical	> 2/3	>2/3
III	Blades	< 2/3	< 2/3
IV	Rodlike	< 2/3	>2/3

Both Zingg [1935] and later Krumbein [194], gave descriptions of riprap shape based on ratios or combinations of the axial dimensions where a, b, and c are, respectively, the major, intermediate, and minor axes; the axes being measured at right angles to each other. Krumbein (7) defines sphericity as the cube root of the ratio of the volume of the circumscribing sphere. This reduces to bc/a^2 . Theoretically, the closer the rock is to a sphere, the closer the b and c dimensions are to being equal to a. This does not take into consideration the possibility of the rock approaching cubical proportions. It is for just such a problem that the Buckingham II-theorem proves useful.

The pragmatic approach to studying a hydraulic system involves such tools as dimensional analysis. Applying the Buckingham II-theorem to the riprap study, the first step is to identify the possible variables. Letting S equal some stability index,

(3.1)

 $S = \phi(D, d, a, b, c, V, s, \theta, n, h_B, y_V, q, \overline{v}, f, W, f_s, f_{A}, \mu)$ where the geometric parameters are identified as:

- d = diameter of a sphere of weight equivalent to the riprap [L]
- a, b, c, = axial dimensions; longest, intermediate, and shortest,

respectively [L]

V =volume of riprap $[L^3]$

s = shape factor [dimensionless]

 θ = position = \emptyset (shape, nesting, etcetera) dimensionless

n = porosity [dimensionless]

hB = drop in energy head across the baffle [L]

where kinematic parameters are identified as:

q = discharge per foot of channel width $[L^3/T/L]$ \vec{v} = mean point velocity [L/T]

f = frequency [1/T]

 y_v = vertical distance from rock surface to significant point

velocity for scour [L]

where dynamic parameters are identified as:

W = weight [F]

 $f_s = \text{density of riprap} \left[\text{FT}^2 / \text{L}^4 \right]$ $f_\omega = \text{density of water} \left[\text{FT}^2 / \text{L}^4 \right]$

h_A = amplitude of the velocity head fluctuations [L]

 μ = dynamic viscosity of the water $[T/L^2]$

According to Buckingham's II-theorem, there must be three repeating variables since there are three fundamental dimensions, length, force
and time present. Usually, a variable is chosen from each of the three categories listed above. In the case of riprap, it would be desirable to have II-terms representing physical features, hydraulic forces, and a combination of the two.

Physical characteristics can be formed into II-terms directly since they are for the most part, a function of length.

- (3.2) $II_1 = \frac{a+b+c}{3d}$
- $(3.3) \qquad II_2 = \frac{a+b+c}{3y_v}$
- $(3.4) \qquad II_3 = V \\ abc$

The third pi term is suggested by the need to compare the rocks! actual shape to a rectangular solid of the same axial dimensions.

The hydraulic force terms are a little more difficult to see without some prior knowledge of dimensionless parameters. Beginning with a familiar ratio, the Froude number, h_B , \overline{v} , and g are used. If $II_{\underline{l}} = A[\overline{v}^{*}, g^{9}, g_{\underline{s}}^{*}, h_{\underline{s}}^{-1}]$ with \overline{v} , g and $f_{\underline{s}}$ the repeating variables, then according to Buckingham's II-theorem:

$$\Pi_{4} = \varphi_{4} \left[(L/T)^{*}, (L/T^{2})^{*}, (FT^{2}/L^{4})^{*}, L^{-1} \right]$$
F: $z = 0$
L: $x + y - hz - 1 = 0$
T: $-x - 2y + 2z = 0$
 $x = 1 - y = -2y$
 $y = -1, \quad x = 2$
(3.5) $\Pi_{4} = \frac{\overline{v}^{2}}{h_{B}}$

For II₅, the last term in equation 3.5 was changed to D ; and g changed to h_B as a repeating variable

$$II_{5} = \phi_{5} \left[(L/T)^{x}, h_{s}^{y}, \rho_{s}^{z} D^{-1} \right]$$

$$II_{5} = \phi_{5} \left[(L/T)^{x}, (L)^{y}, (FT^{2}/L^{4})^{z}, L^{-1} \right]$$

$$F: z = 0$$

$$L: x + y - 4z - 1 = 0$$

$$T: -x + 2z = 0$$

$$x = 2 (0) = 0 \text{ thus } y = 1$$

$$(3.6)$$
 II_c = h_p/D

By inspection;

$$(3.7) \qquad II_6 = \overline{v}^2 \frac{1}{Dg}$$

A combination of hydraulic and physical characteristics can be evaluated by inspection from II_{j} , in equation 3.5:

$$(3.8) \qquad \text{II}_7 = \overline{\boldsymbol{v}}^2/\text{dg}.$$

A number of other II-terms could be developed, but the present study will be limited to these seven. (Tables 15 through 21)

To formulate an equation with the pi terms, another II-term involving S is needed. But since S is defined as an index, it should remain dimensionless. Thus,

(3.9)
$$II_8 = S$$
 and since
(3.10) $S = C \left[II_1 \cdot II_2 \cdot II_3 \cdot II_4 \cdot II_5 \cdot II_6 \cdot II_7 \cdot \cdots \cdot II_{17}\right]$,
where $C = constant$,

the researcher must make some decisions about which terms can be measured and which factors are of negligible consideration for riprap design. Referring back to the list of possible variables, other II-terms to be concerned with, in particular, are those factors which deal with turbulence. The amplitude term (h_A) and frequency term (f) might be linked together in an expression called the scale of turbulence. H. L. Dryden (26) found that a characteristic length was needed in the specification of turbulence. This length was a measure of the magnitude of the turbulent eddies which he called their scale. He found that this length can be experimentally measured by using a correlation coefficient of longitudinal fluctuations u_1 ' and u_2 ' measured at two points whose transverse distance is y. Expressed in equation form:

(3.11)
$$R(y) = \frac{u_1' u_2'}{\sqrt{u_1'^2} \sqrt{u_2'^2}}$$

where u = total velocity fluctation

 \overline{u} = mean velocity fluctuation $u' = u - \overline{u}$ = turbulence fluctuations over and above the mean velocity $\sqrt{u'^2}$ = root-mean-square value of excess turbulence fluctuation.

The quantity which is characteristic of the scale of turbulence is given as:

(3.12)
$$L = \int_0^\infty R(y) dy.$$

It is a measure of the magnitude of the clots of fluid which move together as a unit and thus describe the size of individual eddies. This technique has one difficult requirement for the researcher. It states that y is to varied from zero to infinity, and the root-meansquare value computed for at least a sufficient number of points inbetween. This task would have been out of the question with the

present instrumentation available. Consequently, the scale of turbulence was not taken for the present report.

The idea of a relationship between turbulence and uplift persisted. As the research progressed, the phenomena of a sudden upward thrust was repeated several times. The problem became one of measuring pressure fields down in the riprap and at its surface. The surface profile could be measured with the dual-tube probe, but no way was found to measure pressure among the rocks at the centerline of the channel. The wall ports below the riprap surface were subject to a variety of possible velocity vectors rebounding off of the adjacent rocks as well as just the static pressure head. The water column readings did not fluctuate enough to warrant transducer recording. At any rate, in situations where fluctuations were observed, several readings were taken and averaged. To see if a trend could be established, the data was plotted. From the plottings, Figures 13 through 16, it can be seen that a trend of uplift pressure would be difficult to confirm. All but the smallest size material show the higher pressure for final scour at the port farthest below the rock surface. But this is not the case at the next port above. The pressure was higher above the rock surface for about half the cases of initial instability. This would have to be reversed for uplift to take place. Finally, it is recognized that ports adjacent to rocks below the surface probably do not indicate the true static pressure at these points because of variation in the direction of the velocity vectors around the rocks.

Another point of interest is the profile drag of the material. As was briefly mentioned in the first chapter, rocks should in

general be placed in such a way as to have a minimum number of points projecting significantly above the mean rock surface. The problem of surface drag effects is the reason behind this requirement. Such rocks would be more subject to scour than the others in the blanket. The surface drag magnitude could be established experimentally by rigging one individual rock with a system of spring and levers or even an electrical system which would record its movement. Yet this approach would be difficult to correlate to the overall riprap blanket. Here again is the problem of extreme variation of sample condition where random sampling is recommended.

Attention is now directed back to the factors considered significant and measurable for the present study. Referring to the II-terms again, the following factors are required: a, b, c, d, h_B , \overline{v} , and D. The first four terms have already been discussed since they could be measured before flow tests in the flume could begin. The last three terms all concern the velocity jet at the boundary of the riprap surface.

The h_B term represents the drop in head across the baffle. The energy on either side of the baffle is distributed as shown in the equation 3.13 for a unit width section.

(3.13) $y_1 + \frac{v_1^2}{2g} = y_2 + \frac{v_2^2}{2g} + h_1$

where $y_1 \& y_2$ = upstream and downstream depth $v_1 \& v_2$ = upstream and downstream velocity h_1 = energy loss at the baffle

If the unit width areas of flow for upstream and downstream are assumed to be y_1 and y_2 , respectively, then the continuity equation

can be stated:

$$(3.14) \qquad q = v_1 y_1 = v_2 y_2 .$$

Combining equations 3.13 and 3.14, and solving for vo,

(3.15)
$$v_2 = \frac{c_v \sqrt{2g(y_1 - y_2)}}{\sqrt{1 - (y_2/y_1)^2}}$$

The corresponding flow rate can then be stated as

(3.16)
$$q = \frac{c_v c_c A}{\sqrt{1 - (y_2/y_1)^2}} \sqrt{\frac{2g(y_1 - y_2)}{2g(y_1 - y_2)}}$$

where c_v accounts for the energy loss, h_1

c accounts for the contraction of the submerged

jet.

The depths were measured with stilling well manometers attached to static ports located at the base of the riprap blanket. Standard hook type point gauges were used to measure the depth of the water relative to the base of the riprap layer on the flume floor. The respective depth readings of the manometer wells were related by using a standard Wye Level to establish a level line. The distance from the flume floor was also established with a level rod accurate to 0.01 foot. Before continuing, it seems pertinent to discuss the aspect of accuracy as it pertains to this research effort.

Validity of results in research depends heavily on the judgement shown in evaluating the accuracy of data measurements. Extreme care must be exercised in some very delicate measurements such as those made at the point of initial instability, while other measurements may be by necessity, rather arbitrarily taken. For example, the stilling well gauges may be read to the nearest 0.001 foot. Water surface elevation for undulating flow may be good to the nearest 0.1 foot with some reservation for the time of recording. To add to the problem, the backwater effect must be observed. To accomplish a realistic set of data, the stilling well measurements were read at least before and after the velocity profile was taken and sometimes more frequently as the circumstance warranted. The observed trend can be described as an asymptotic approach to a constant depth achieved upon the return to the steady state or equilibrium force condition. This return time for the present system was short enough that by the time a vertical velocity traverse of the jet was made, the flow system was sufficiently stabilized that the depth readings then recorded could be safely used. The difference $y_1 - y_2$ was used as the parameter h_p .

The boundary of the submerged jet was made evident by air bubbles in the flow. The jet contained very few if any air bubbles, whereas the turbulent portion above it contained many bubbles which stood out very well when illuminated by flood lamp (Figure 4B). The fluid interface was traced on the glass wall in order to estimate how much jet contraction existed. The coefficients c_V and c_C were assumed to be combined in a single term c. This coefficient, then, is equal to the ratio of the vertical dimension from the flume floor to the fluid interface, divided by the vertical distance of the baffle edge above the floor (Figure 1). The cross section was roughly plotted by eye using vertical measurements from the baffle edge to the rock surface at three inch intervals. Subtracting the mean riprap blanket thickness from the height of the contracted jet boundary gives the parameter D. This information was all measured for the tests following: 10D, 11B, 11D, 12D, 1hB, 15B, 16B, 19B, 20B. If tests can be continued

over a longer time period, the initial instability test and the final scour test will be run separately, pausing between to allow the flume to drain so that the cross section of the initial instability surface can be recorded.

Using equation 3.16, the above measurements and the discharge measured by the Venturi meter, the total flow under the baffle can be checked. By integrating the vertical velocity traverse with a planimeter and dividing by D, the mean velocity of the jet is determined. If the area of the jet is assumed to be that at the recorded cross section and determined by planimeter, then the flow in the jet is represented by the equation

 $(3.17) q_j = \overline{v}_{mean} A_c,$

where $A_c = contracted$ area of flow.

The difference of the total flow minus the jet flow should provide a fair estimate of the water moving through the riprap. This latter flow should in turn give some indication of the relative stability according to size and shape categories.

The velocity profiles must be investigated next to see if a significant velocity exists. The velocity in the profiles represents a condition which results after the forces causing instability reach equilibrium. Consequently, the velocity recorded for both instability and final scour for each test represents a span of values from slightly less than instability velocity to slightly less than the final scour velocity. The measurements at each point represent an . average value of the velocity head at that point. Velocity head fluctuations were highest at the boundaries, both at the riprap surface and at the water interface between the jet and the backwash,

from the submerged hydraulic jump downstream from the test section (Figure 4B). The shape or divergence angle of the upper and lower portions of the velocity traverse were affected by a change in the tailwater condition and by discharge variation. In attempting to find a significant velocity most affecting scour, several tests were run for the same size riprap at different discharges. Curves from the different discharges are compared in Figures 19, 20, 21, 22, 24, and 25. Final scour was hard to duplicate. As a result of this fact plus the different berm formation of scoured material downstream, the position of the curve is shifted with respect to the datum. Consequently, there is quite a difference in the shape and maximum velocity for the final scour curves. A good example is found in Tests 20 and 21 where the final scour was different due to the existence of a slightly higher differential head across the baffle in Test 20. This resulted in a higher maximum velocity than in Test 21. There was a slightly higher berm formed in Test 20 downstream from the velocity probes elevating the velocity curve relative to the assumed datum. Both tests were conducted for conditions approaching the extreme capacity of the flume allowing little leeway for adjustment. During both tests, the downstream wheels (Figure 2) of the carriage holding the movable baffle were forced into the air by the tremendous force of the water on the baffle.

Special note should be made of the riprap thickness indicated in Figures 24 - 29. Theoretically, the thickness of the rock blanket should not affect the velocity it takes to scour the blanket to a depth that is less than riprap depth. In the current study, however, the small material used to help block flow through the riprap was

not present. As a result, the profile drag was not as great and a higher velocity was required for movement (see Test 15A, Figure 24).

The use of the double probe in Tests 17 through 21 gives a more accurate picture of the rapid fluctuations at a given time. By spacing the two Prandtl tubes in such a way as to duplicate velocity measurements in the center region of the jet, a check was made on the data from each separate tube.

The velocity significant for the initial instability was difficult to establish. Point velocities selected for study included the maximum jet velocity, the mean jet velocity, and the velocity at an arbitrary point which usually was placed at an abrupt change in the velocity gradient. This last point was signified by the sign, **4**. Point plots based on maximum jet velocity and the so called arbitrary velocity best fit the envelope conditions shown in Figures 32 and 33 when the average diameter per category was used. (Tables 13 and 14)

Finally, in order to establish some model scale for size projections to larger riprap material, the II-terms must be developed further. Figure 37 gives the general variation of II-terms with the different sizes and shapes tested.

Avaluation Outline of Tests by Mumber Taken From Daily Reports

1A. Good shape. Too low discharge for size of rock. Too much influence per rock on velocity profile. Difficult to duplicate scour test.

1B. Not plotted since velocity probe was not at the maximum contraction or throat of the jet.

2A and 2B. No Test 2A was run. Actual sequence follows: Tests 1A and 1B were run. Prandtl tube moved upstream to throat of jet. Discharge slowly built up to same as that of Test 1. Profile 2B then taken over scour of Test 1B.

3A. Good correlation with similar test, 2B. Flow in rocks prevents profile from returning to zero at rock surface.

4A. Fairly good correlation with lower discharge in 3A. With higher discharge, larger baffle opening provided needed velocity, thus not as dependent on each rock's position at the centerline:

4B. Not run.

5A. Good profile.

5B. Trouble with rock berm near probe. Definitely affected shape of curve.

6A. Good profile. Maximum velocity reduced by low discharge.

6B. Needs more data points in critical gradient either side of maximum velocity.

7A. Good profile. Good correlation with high discharge profile in Test 8A.

7B. Good profile. Some scatter for maximum velocities due to rock #4 projecting into current directly under probe.

SA. Best profile for 3 to 6 inch cobbles for initial instability.

. 53. Good profile. Rocks numbered 1 and 11 interfered with probe reading in scour.

9A and 9B. Not plotted. Discharge too low even to move 6 to 9 inch cobbles for scour test.

10A and 10C. Good agreement. Appears to be slight shift in datum. 10B and 10D. Slightly higher maximum velocity on Test 10D. Test 10B should have more measurements in region of maximum velocity, but good profile at riprap surface showing velocity gradient.

11A and 11C. Same as 10A and 10C.

11B and 11D. Good agreement in maximum velocities.

12A, 13A, 14A, 15A, 16A, 17A. Good general agreement in all but Test 15A where initial scour data was questionable. Movable baffle would not raise high enough for twenty-two second-foot discharge resulting in some instability as this discharge was being obtained. , 12B, 13B, 14B, 15B, 16B, 17B. Good agreement in all but lower discharge tests, numbers 16B and 17B. For Test 16B, rock number 15 affected lower profile data.

18A and 19A. Slight shift in velocity magnitude. Also vertical translation of profile. Cause traced to difference in moved rock formation.

18B and 19B. Good agreement for final scour tests.

20A and 21A. Largest material tested. Almost too big for flume. Profile shape definitely affected by individual rock position.

20B and 21B. Same as 20A and 21A.

Conclusions and Recommendations

The choice of the diameter affected the plot in Figures 32 and 33 noticeably. The U.S. Army Corps of Engineers' curves (Figure 35) are based on equivalent diameter as developed in equations 1.2 and 1.4. The weight used in finding the equivalent diameter is plotted parallel to the diameter. The diameter also represents the fifty per cent passing size of stone as measured in a sieve analysis. V_m represents the mean velocity and Vs, the velocity against the stone. The U.S. Eureau of Reclamation (Figure 34) recommends that ... "most of the stone should be of the size indicated by the curve (14)." The U.S. Bureau of Reclamation also uses the equivalent diameter and weight for parallel ordinates. The velocity used is an estimated bottom velocity. Andersson's curves (Figure 36) are based on the equations as outlined on pages 10 and 11. The curve envelopes of Figures 32 and 33 represent higher values of velocity for failure of a given size. The writer believes this difference to be due to the lack of graded material filling the voids in the material tested. Since the curves obtained from other sources all represent some form of gradation, a true comparison could not be made until a sieve analysis was made available on a random sample from each gradation represented.

Estimates of discharge in the gravel proved to be difficult Differences in the estimated and actual effective cross section are believed to be the reason that, for instance, the discharge in the jet for Test 19 was greater than the total measured discharge in the flume (Tables 22 and 23). The discharge computed on the basis of equation 3.16 is too high because the correct value of c_v was never estimated. A commuter program entitled "AREVEL" was used to commute the information (Tables 22 and 23). This program is found in the Appendix along with other computer programs used to calculate shape characteristics and II-terms for physical features.

The plot of the pi terms proved indecisive. Hore rock sizes are needed to provide any sort of trend. For the angular material, the v^2/dG terms vary inversely with the size since d or the equivalent diameter is in the denominator. The v^2/dG varies directly with the riprap size. The depth of the jet is reduced as the velocity is increased by moving the baffle down while holding discharge constant. This term reflects the special conditions imposed on this specific system used for tests and does not show promise for prototype applications. The term hp/D varies directly with the material size. This trend is expected since the differential head must increase across the baffle as the velocity increases. This increase in h_R combined with the previously discussed decrease in D accounts for the increase in hp/D. This term, too, is restricted for use in the present study system of a submerged jet under a baffle. The II-terms representing the physical characteristics found in Tables 15 through 20 are difficult to use in their present form. They represent values for each individual rock per random sample per category. As such they must be lumped together in such a way as to best represent the category in question. A form of statistical analysis such as standard deviation would seen appropriate. This part of data reduction is recommended for the first chase of the overall study of rinran. The term "model study" can only be applied to this study in a general sense in that the rock sample in the flume did not similate prototype riprap.

A basic research project of this type is merely a first sten in a series of studies aimed at improving riprap design criteria. The next step after the scale projection of size will be tests to determine optimum gradation for the general case of a submerged jet such as used in this study. Next, a series of tests on different specific situations would be appropriate. For example, the jet which occurs downstream from a chute with no energy dissipator other than riprap could be studied by building a deck under the baffle and lowering the tailwater conditions to increase velocity. The velocities resulting should be high enough to provide data from the present level flume system.

In conclusion, the writer recommends that any approach to the problem of riprap should be made with a research program based on principles of random selection of data. This data should be systematically analysed by methods of modern statistical techniques made. possible by the use of the new high speed digital computors. For the mass of data especially concerning the velocity profiles and turbulent pressure fluctuations, a direct system of digitized tape recording as well as line recordings would be very useful. Then data could be directly entered into the computor for analysis.

The present study is considered by the writer to be a meaningful beginning. The velocities in general are a valid representation of the conditions defined by the study. The initial instability velocities are believed to be more reliable than the final scour velocities. . Hore rock sizes will need to be tested before this phase of the general study can be considered complete.

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PHYSICAL CHARACTERISTICS OF RIPRAP 3/4 TO 1 1/2 INCH COBBLES

005	REM	AXIAL	DIMENSI	ONS(IN)	WΤ	VØL
007	REM	A	В	С	LBS	ML
010	DATA.	2.125,	1.875,	1.125,	.24,	44
020	DATA	2.125,	1.875,	.875,	.17,	30
030.	DATA	2.250,	1.625,	.625,	.14,	25
040	DATA	2.500,	1.625,	.750,	.16,	28
050	DATA	1.250,	1.000,	.750,	.04,	9
060	DATA	2.000,	1.500,	1.000,	.16,	30
070	DATA	2.125,	.1.375,	1.125,	.17,	30
080	DATA	1.875,	1.750,	1:125,	.20,	34
090	DATA	2.125,	1.250,	1.125,	.14.	25
100	DATA	1.500,	1.250,	1.125,	.09,	15
110	DATA	1.375,	1.250,	1.000,	.08,	14
120	DATA	1.625,	1.750,	1.000,	.11,	20
130	DATA	1.250,	1.000,	.500,	•03;	4
140	DATA	2.000,	1.625,	. 625,	.13,	22
150	DATA	1.125,	1.000,	.750,	.04,	7
160	DATA	1.375,	1.000,	.875,	.05,	10
170	DATA	1.750,	1.250,	.625,	.07,	6
180	DATA	1.875,	1.625,	1.000,	.19,	27
190	DATA	2.500,	1.625,	1.000,	.17,	30
200	DATA	2.000,	1.750,	.750,	.18,	27
210	DATA	1.375,	1.000,	1.000,	.07,	11
220	DATA	2.125,	1.500,	1.000,	.18,	28
230	DATA	1.875,	1.375,	.750,	•10,	55
240	DATA	2.250,	1.625,	1.125;	.553	41
250	DATA	1.375,	1.250,	1.000,	•09 •	8
260	DATA	1.500,	1.000,	1.000,	.07,	11
270	DATA	2.250,	1.375,	1.000.	•17,	34
280	DATA	2.000,	1.500,	1.250,	.17,	30
290	DATA	2.000,	1.500,	.875,	.13,	24
300	DATA	1.875,	1.500,	1.125,	• 16 •	27
310	DATA	5.000,	1.625,	.875,	• 11 =	23
350	DATA	2.625,	1.500,	1.000,	.18,	40
. 330	DATA	1.500,	1.375,	1.000,	•09,	18
340	DATA	2.125,	1.500,	1.250,	.19,	39
, 350	DATA	2.375,	1.500,	1.125,	• 16 :	28
1360	DATA	2.375,	1.500,	1.375,	.25,	44

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PHYSICAL CHARACTERISTICS OF RIPRAP 1 1/2 TO 3 INCH COBBLES

005	REM	AXIAL D	IMENSIØ	NS(IN)	WT	VØL
007	REM	A	В	С	LBS.	ML
010	DATA	3.000,	2.375,	1.625,	.56,	95
'020	DATA	4.500,	3.875,	1.875,	1.57,	265
030	DATA	3.750,	2.875,	1.625,	.81,	135
040	DATA	2.625,	2.125,	1.000,	.33,	62
050	DATA	3.000,	1.750,	1.750,	. 42,	80
060	DATA	3.375,	2.500,	1.625,	.67,	120
070	DATA	3.000,	2.500,	1875,	•69,	1.22
080	DATA	3.250,	1.875,	1.625,	• 41,	74
090	DATA	2.750,	2.000,	1.375,	. 43,	75
100	DATA	3.250,	2.750,	1.125,	.56,	100
110	DATA	3.000,	2.000,	1.750,	• 49,	90
120	DATA	3.250,	2.375,	1.250,	.59,	105
130	DATA	4.000,	.2.250,	2.000,	•84,	145
140	DATA	4.000,	3.250,	1.500,	1.17,	230
150	DATA	3.500,	2.250,	1.375,	.54,	95
160	DATA	4.375,	2.750,	1.000,	.59,	105
170	DATA	2.500,	2.125,	1.250,	.33,	64
180	DATA	2.750,	2.500,	1.750,	.52,	95
190	DATA	2.500,	1.875,	1.000,	.28,	52
200	DATA	4.000,	3.000,	1.750,	1.12,	210
210	DATA	2.625,	1.875,	1.250,	•34,	60
220	DATA	4.750,	2.000,	1.875,	•91,	160
230	DATA	3.000,	2.750,	1.125,	• 48,	88
240	DATA	3.125,	2.250,	1.500,	•66,	120
250	DATA	2.000,	1.750,	1.250,	.24,	45
260	DATA	2.375,	2.000,	1.250,	•26.	48
270	DATA	2.000,	1.625,	1.625,	•26,	50
280	DATA	2.250,	1.750,	1.125,	.23,	42
290	DATA	5.000,	3.375,	1.750,	1 • 49 ,	260
300	DATA	3.875,	3.000,	2.000,	1.39,	250
310	DATA	3.500,	2.625,	2.000,	1.04.	180
320	DATA	3.500.	2.125,	5.0003	.77,	130
330	DATA	2.875,	1.750,	1.000,	.27,	52
340	DATA	2.750,	2.250,	1.500,	• 48,	84
350	DATA	2.750,	1.875,	1.125,	26,	48
360	DATA	2.500,	1.875,	1.125,	.24,	45

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PHYSICAL CHARACTERISTICS OF RIPRAP A. 3 TO 6 INCH COBBLES

					1100 000		
005	REM	AXIAL	DIMENSI	ONS (IN:) WEIGH	T VE	LUME
007	REM	A	В	С	LBS	5	ML
010	DATA	5.625,	5.375,	1.375,	2.325,	400	
020	DATA	6.875,	4.375,	2.625,	3.053,	530	
030	DATA	7.625,	5.375,	3.000,	5.578,	890	
040	DATA	6.000,	3.500,	2.500,	2.470,	315	
050	DATA	7.375,	3.250,	3.250,	4.395,	510	
060	DATA	6.000,	4.750,	3.625,	4.370,	690	
070	DATA	4.750,	3.875,	1.375,	1.610,	220	
080	DATA	5.500,	3.875,	1.500,	1.678,	220	
090	DATA	6.000,	4.375,	2.625,	3.785,	530	
100	DATA	5.625,	5.375,	1.375,	2.325,	400	
110	DATA	6.625,	4.750,	3.250,	4.705,	695	
120	DATA	7.625,	5.750,	3.500,	7.515,	1170	i la compañía de la c
130	DATA	5.500,	4.000,	2.625,	2.600,	322	
1.40	DATA	6.125,	3.875,	2.000,	2.148,	420	
150	DATA	4.750,	4.125,	2.625,	2.371,	250	
160	DATA	6.375,	6.125,	4.250,	7.277,	1100	
170	DATA	6.938,	6.500,	2.625;	5.355.	730	

B. 6 TO 9 INCH CUBBLES

	D. 01				
REM	AXIAL DI	MENSIONS	S (IN)	WEIGHT	VØLUME
REM	A	B·	С	LBS	ML
DATA	11.750,	10.000,	4.500,	23.08,	3840
DATA	11.500,	8.000,	5.500,	25.02,	4230
DATA	9.250,	8.750,	5.125,	21.01,	3480
DATA	9.125,	8.750,	5.875,	23.75,	3840
DATA	8.500,	6.750,	5.250,	13.44,	2380
DATA	11.000,	7.500.	5.500,	23.69,	3950
DATA	11.500,	8.500,	5.625,	23.46,	4140
DATA	8.750,	7.750,	6.500,	20.42,	3445
DATA	9.500.	7.875,	4.875,	14.75,	2425
DATA	9.250,	7.875,	4.000,	12.65,	2090
DATA	11.375,	8.750,	4.500,	19.72,	3340
DATA	11.625,	7.875,	4.000,	15.84,	2730
DATA	10.875.	9.375,	4.500,	19.62,	3340
DATA	10.750,	6.625,	4:625,	15.19,	2530
DATA	8.000,	7.000,	6.625,	. 15.68,	2620
DATA	10.750,	5.625,	4.000,	12.47,	2060
DATA	10.375,	7.375,	5.125,	13.61,	2260
DATA	10.750;	6.750,	5.875,	14.45,	3130
	REM REM DATA DATA DATA DATA DATA DATA DATA DAT	REM AXIAL DI REM A A DATA 11.750, DATA 11.500, DATA 11.500, DATA 9.250, DATA 9.250, DATA 9.125, DATA 9.125, DATA 9.125, DATA 9.125, DATA 9.125, DATA 9.125, DATA 9.125, DATA 9.125, DATA 10.00, DATA 11.000, DATA 11.000, DATA 11.000, DATA 11.000, DATA 9.500, DATA 9.250, DATA 9.250, DATA 10.375, DATA 11.375, DATA 11.375, DATA 10.750, DATA 10.750, DATA 10.750, DATA 10.750, DATA 10.375, DATA 10.750,	REM AXIAL DIMENSIONS REM A B DATA 11.750, 10.000, DATA 11.750, 10.000, DATA 11.500, 8.000, DATA 9.250, 8.750, DATA 9.125, 8.750, DATA 11.000, 7.500, DATA 11.000, 7.500, DATA 11.500, 8.500, DATA 11.500, 8.500, DATA 9.500, 7.875, DATA 9.250, 7.875, DATA 9.250, 7.875, DATA 11.375, 8.750, DATA 11.375, 8.750, DATA 11.625, 7.875, DATA 10.875, 9.375, DATA 10.750, 6.625, DATA 10.750, 5.625, DATA 10.375, 7.375, DATA 10.750, 6.750, </td <td>REM AXIAL DIMENSIONS (IN) REM A B C DATA 11.750, 10.000, 4.500, DATA DAT</td> <td>REM AXIAL DIMENSIONS (IN) WEIGHT REM A B C LBS DATA 11.750, 10.000, 4.500, 23.08, DATA 11.500, 8.000, 5.500, 25.02, DATA 11.500, 8.000, 5.500, 25.02, DATA 9.250, 8.750, 5.125, 21.01, DATA 9.125, 8.750, 5.875, 23.75, DATA 9.125, 8.750, 5.250, 13.44, DATA 9.125, 8.750, 5.250, 13.44, DATA 11.000, 7.500, 5.500, 23.69, DATA 11.000, 7.500, 5.625, 23.46, DATA 11.000, 7.500, 5.625, 23.46, DATA 11.000, 7.500, 5.625, 23.46, DATA 11.600, 7.750, 6.500, 20.42, DATA 11.500, 8.500, 5.625, 23.46, DATA 12.65, DATA 9.500, 7.875, 4.000, 12.65, DATA DATA 9.250, 7.875, 4.000, 12.65, DATA DATA 9.250, 7.875, 4.000, 15.84, DATA DATA 10.875, 9.375, 4.500, 19.62, DATA DATA 10.875, 9.375, 4.500, 19.62, DATA DATA 10.750, 6.625, 4.625, 15.19, DATA DATA 10.750, 5.625, 4.000, 12.47, DATA</td>	REM AXIAL DIMENSIONS (IN) REM A B C DATA 11.750, 10.000, 4.500, DATA DAT	REM AXIAL DIMENSIONS (IN) WEIGHT REM A B C LBS DATA 11.750, 10.000, 4.500, 23.08, DATA 11.500, 8.000, 5.500, 25.02, DATA 11.500, 8.000, 5.500, 25.02, DATA 9.250, 8.750, 5.125, 21.01, DATA 9.125, 8.750, 5.875, 23.75, DATA 9.125, 8.750, 5.250, 13.44, DATA 9.125, 8.750, 5.250, 13.44, DATA 11.000, 7.500, 5.500, 23.69, DATA 11.000, 7.500, 5.625, 23.46, DATA 11.000, 7.500, 5.625, 23.46, DATA 11.000, 7.500, 5.625, 23.46, DATA 11.600, 7.750, 6.500, 20.42, DATA 11.500, 8.500, 5.625, 23.46, DATA 12.65, DATA 9.500, 7.875, 4.000, 12.65, DATA DATA 9.250, 7.875, 4.000, 12.65, DATA DATA 9.250, 7.875, 4.000, 15.84, DATA DATA 10.875, 9.375, 4.500, 19.62, DATA DATA 10.875, 9.375, 4.500, 19.62, DATA DATA 10.750, 6.625, 4.625, 15.19, DATA DATA 10.750, 5.625, 4.000, 12.47, DATA

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PHYSICAL CHARACTERISTICS OF RIPRAP 1 1/2 TO 3 INCH ANGULAR RIPRAP

005	REM	AXIAL	DIMENSION	S(IN)	WT	H ØF	CYL VOL
007	REM	A	В	C	. LBS	H1	H2
010	DATA	4.375,	2.500,	2.000,	1.23,	1.910,	1.925
020	DATA	4.750,	3.000,	2.000,	1.46,	1.910,	1.926
030	DATA	6.000,	2.000,	1.750,	1.35.	1.910,	1.926
040	DATA	4.250,	2.625,	1.000,	.54,	1.909,	1.916
050	DATA	5.500,	3.000,	2.750,	1.72,	1.909,	1.925
060	DATA	6.000,	2.875,	2.875,	1.94,	1.908,	1.928
070	DATA	5.000,	2.750,	2.750,	1.38,	1.908,	1.920
080	DATA	5.500,	3.250,	1.500,	1.00,	1.907,	1.918
090	DATA	5.000,	3.000,	2.750,	1.78,	1.907,	1.926
100	DATA	4.500,	3.000,	1.625,	1.50,	1.907,	1.922
110	DATA	4.750,	2.875,	2.500,	1.28,	1.906,	1.919
120	DATA	5.500,	3.750,	1.250,	1.26,	1.906,	1.919
130	DATA	6.250,	2.750,	2.500,	2.09,	1.906,	1.927
140	DATA	6.250,	3.000,	2.750,	2.24,	1:906,	1.928
150	DATA	4.500,	3.500,	2.500,	1.27,	1.905,	1.918
160	DATA	4.000,	3.250,	1.500,	• 90 •	1.904,	1.917
170	DATA	6.000,	3.000,	2.500,	2.02,	1.904,	1.924
180	DATA	6.500,	2.750,	2.000,	2.05,	1.904,	1.927
190	DATA	4.500,	2.500,	2.500,	1.9.4.	1.09040	1.919
200	DATA	5.000,	2.250,	1.625,	1.43,	1.904,	1.919
210	DATA	4.000,	3.000,	3.000,	1.30,	1.903,	1.918
220	DATA	5.000,	2.375,	1.250,	.54,	1.903,	1.909
230	DATA	6.250,	3.500,	1.500,	1.42,	1.903,	1.918
240	DATA	5.500,	2.500,	2.500,	2.63,	1.903,	1.920
250	DATA	4.750,	3.750,	2.000,	1.35,	1.902,	1.916
260	DATA	7.750,	3.500,	1.750,	2.01,	1.902,	1.930
270	DATA	5.500,	3.500,	1.750,	175,	1.902,	1.922
280	DATA	5.250,	3.500,	2.250,	1.87,	1.901,	1.921
290	DATA	4.500,	2.750,	2.625,	1.80.	1.901;	1.920
300	DATA	8.250,	2.500,	1.750,	1.56,	1.901,	1.918
310	DATA	5.750,	3.000,	2.500,	1.81,	1.900,	1.920
320	DATA	6.500,	3.250,	2.000,	2.08,	1.900,	1.922
330	DATA	5.500,	2.875,	2.250,	2.15,	1.900,	1.923
340	DATA	4.500,	3.500,	1.250,	1.06,	1.900,	1.910
350	DATA	5.250,	1.750,	1.375,	.97,	1.900,	1.910
360	DATA	5.250,	3.250,	1.500,	1.11,	1.900,	1.911
370	DATA	4.500,	3.000,	2.250,	1.43,	1.899,	1.914
380	DATA	5.000.	3.500,	2.000,	1.33,	1.899;	1.913
390	DATA	5.000,	2.625,	2.500,	1.28,	1.899,	1.913

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PHYSICAL CHARACTERISTICS OF RIPRAP 3 TO 6 INCH ANGULAR

005	REM	AXIAL	DIMENSI	ØNS(IN)	WT	VØL
007	REM	A	В	С	LBS	ML
010	DATA	8.750,	6.000,	3.750,	6.06,	980
020	DATA	5.000,	4.250,	2.000,	1.62,	250
030	DATA	7.000,	4.250,	3.750,	4.94,	880
040	DATA	9.000,	4.000,	3.250,	6.78,	1160
050	DATA	5.500,	3.500,	3.250,	1.71,	300
060	DATA	9.250,	5.750,	2.500,	5.10,	860
070	DATA	7.500,	5.250,	3.375,	5.86,	1000
080	DATA	7.000,	4.500,	3.750,	4.55,	760
090	DATA	7.000,	4.000,	2.000,	2.08,	320
100	DATA	5.000,	4.000,	3.250,	3.30,	540
110	DATA	5.000,	4.375,	4.000,	4.01.	660
120.	DATA	5.000,	3.875,	2.500,	1.46,	200
130	DATA	9.375,	5.000,	4.000,	8.05,	1310
140	DATA	5.500,	5.375,	.3.125,	4.01,	560
150	DATA	8.000,	4.750,	3.375,	5.08,	800
160	DATA	7.125,	4.625,	3.750,	4.51,	750
170	DATA	7.125,	6.000,	2.000,	3.59,	. 570
180	DATA	8.000,	4.625,	3.375,	5.00,	840
190	DATA	5.375,	4.500,	2.500,	2.20,	420
500	DATA	6.750,	4.750,	3.000,	4.47,	730
210	DATA	9.000,	4.500,	2.750,	5.02,	720
550	DATA	7.500,	4.625,	3.875,	5.80,	960
230	DATA	5.375,	4.75,	3.250,	3.62,	560
240	DATA	5.500,	3.000,	2.500,	2.77,	440
250	DATA	5.250,	4.000,	4.000,	2.95,	530
260	DATA	7.000,	5.125,	3.250,	4.76,	830
270	DATA	6.500,	4.500,	3.875,	3.74,	660
280	DATA	4.750,	3.125,	3.125,	1.88,	320
290	DATA	6.000,	4.375,	3.375,	3.87,	650
300	DATA	7.500,	5.000,	3.500,	5.63.	920
310	DATA	5.750,	3.000,	2.000,	1.82,	350
320	DATA	6.000,	3.750,	2.750,	2.95,	480
330	DATA	6.000,	5.000,	3.500,	3.04,	730
340	DATA .	5.750,	4.000,	3.750,	5.31,	880
350	DATA	5.375.	3.000,	1.500.	.94,	180
360	DATA	9.500.	5.500,	3.875.	6.58.	1020

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PHYSICAL CHARACTERISTICS OF RIPRAP 6 TO 9 INCH ANGULAR

005	REM	AXIAL	DIMENSIØ	VS (IN)	WEIGHT	HT ØF	EO CYL
007	REM	A	В	. C	LBS	H1	H2
010	DATA	12.500,	7.250,	7.250,	40.14,	1.420,	1.836
020	DATA	8.750,	6.250,	5.000,	19.56,	1.418,	1.616
030	DATA	12.500,	9.250,	7.250,	29.40,	1.418,	1.780
040	DATA	12.000,	8.250,	6.125,	25.71,	1.415,	. 1.682
050	DATA	12.750,	8.125,	5.375,	25.92,	1.415,	1.685
060	DATA	13.750,	8.000,	7.250,	26.93,	1.414,	1.700
070	DATA	10.750,	10.375,	5.625,	27.12,	1.412,	1.698
080	DATA	8.500,	7.375,	5.000,	19.68,	1.411,	1.617
090	DATA	9.250,	8.750,	7.750,	31.47,	1.410,	1.730
100	DATA	10.375,	7.250,	5.875,	22.55,	1.408,	1.641
110	DATA	9.625,	7.750,	6.125,	26.82,	1.406,	1.694
120	DATA	11.500,	7.750,	6.875,	24.50;	1.404,	1.651.
130	DATA	11.250,	8.625,	5.000,	27.51,	1.400,	1.678
140	DATA	9.250,	6.750,	5.750,	16.68,	1.405,	1.582.
150	DATA	13.500,	6.375,	6.125,	20. 62,	1.405,	1.622
160	DATA	14.750,	8.250,	7.250,	45.37,	1.420,	1.889
170	DATA	14.000,	9.000,	7.125,	35.56,	1.564,	1.927
180	DATA	9.500,	9.500,	7.250,	32.77,	1.555,	1.900
190	DATA	9.500,	6.125,	4.125,	16.50,	1.566,	1.740
200	DATA	10.250,	7.750,	5.994,	25.10,	1.563,	1.820
210	DATA	11.000,	8.000,	6.750,	39.79,	1.515,	1.915
220	DATA	11.250,	9.000,	8.750,	33.93,	1.508,	1.859
230	DATA	10.750,	7.750.	5.750,	18.81,	1.502,	1.697
240	DATA	9.000,	6.625,	6.500,	24.76,	1.500,	1.761
250	DATA.	14.250,	10.000,	4.750.	25.63,	1.499,	1.767
260	DATA	10.500,	7.000,	7.000,	22.12.	1.502,	1.737
270	DATA	8.500,	8.250,	6.250,	17.66,	1.499,	1.685
280	DATA.	12.750,	8.125,	6.250,	27.84,	1.488,	1.777
290	DATA	10.000,	10.000,	5.750,	23.26,	1.485,	1.724
300	DATA	10.000,	9.500;	8.000,	27.02,	1.484,	1.765
.310	DATA	17.000,	9.125,	7.375,	27.28,	1.484,	1.765
320	DATA	9.250,	7.750,	7 .000,	21.52.	1.482,	1.705
330	DATA	10.125,	9.125,	4.500,	18.81,	1.473,	1.666
340	DATA	13.750;	8.750,	6.000,	26.86,	1.470,	1.753
.350	DATA	10.000,	8.250,	7.000,	18.03.	1.468,	. 1.650
360	DATA	14.250,	7.750,	4.875,	24.52,	1.468,	1.726

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Sec. 14

TABLE 7A

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SHAPE CHARACTERISTICS 3/4 TO 1 1/2 INCH COBBLES

ZING CLASSIFICATIONS: PER CENT PER CATEGORY I II III IV DISK SPHERICAL BLADE RØDLIKE 8.33333 16.6667 41.6667 33.333

RØCK			ZING		BC/At2	•	VØL	
NUMBER	2		CLASS		SPHERICIT	ſΥ	CU FT	
1			. III		. 467128	1	1.55364	E-3
2			III		•363322		1.05930	E-3
3			III		•200617		8.82750	E-4
4			I		•195	2	9.88680	E4
5			IV		• 48		3.17790	E-4
6			III		.375		1.05930	E-3
7			II		• 342561		1.05930	E-3
8			. III		.56		1.20054	E-3
9			II		•311419		8.82750	E-4
10			IV		.625		5.29650	E-4
11			IV		.661157		4.94340	E-4
12			III		.662722		7.06200	E-4
13			III		• 32		1.41240	E-4
14			III		.253906		7.76820	E-4
15			IV		.592593		2.47170	E-4
16			IV		.46281		3.53100	E-4
17			III		.255102		2.11860	E-4
18			III		• 462222		9.53370	EZ ·
19			I		.26		1.05930	E-3
20			III		.328125		9.53370	E-4
		- 100 10	住口 しつ ひい イー 特日	12 - 12 - 12 - 12 - 12 - 12 - 12 - 12 -				

TABLE 7B

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SHAPE CHARACTERISTICS 3/4 TO 1 1/2 INCH COBBLES (CON'T)

RØCK		ZING		BC/At2 .	VØL	
NUMBER		CLASS		SPHERICITY	CU FT	
21		IV		.528926	3.88410	E-4
22		III		• 33218	9.88680	E-4
23		III		•293333	7.76820	E-4
24		IV	1.5	•361111	1.44771	E-3
25		IV		.661157	2.82480	E-4
26		II		• 444444	3.88410	E-4
27		II		.271605	1.20054	E-3
28		IV		• 46875	1.05930	E-3
-29		III		• 328125	8.47440	E-4
30	1.2	IV		• 48	9.53370	E-4
31		III ·		• 355 469	8.12130	E-4
32		I		.217687	. 1.41240	E-3
33		IV		•611111	6.35580	E-4
34		IV		• 415225	1.37709	E-3
35		II		.299169	9.88680	E-4
36		II		• 365651	1.55364	E-3

S PHERICI	TY	CLA	ISSI	FIC	ATI	ØN =	E C*B	1/A12
PERCENT-	0	.0 1	0 0) • 1 =	= • 0			
PERCENT-	0	.17	0 0	.2 :	= 2	.777	78	
PERCENT	-0	.2 1	ØC).3 :	= 2	2.22	222	
PERCENT	-0	.3 1	0 0) • 4 :	= 3	0.55	556	
PERCENT	-	0.4	ΤØ	0.5	=	22.	2222	
PERCENT	-	0.5	TØ	0.6	=	8.3	33333	
PERCENT	-	0.6	TØ	0.7	=	13.	8889	
PERCENT	-	0.7	TØ	0.8	=	0	•	
PERCENT		0.8	ΤØ	0.9	=	0		
PERCENT	-	0.9	ΤØ	1.0	=	0		

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TABLE 8A

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SHAPE CHARACTERISTICS 1 1/2 TO 3 INCH COBBLES

ZING	CLAS	SIFI	CATIØNS:	PER	CENT	PER	CATEGØRY	
I			II		I	II		IV
DISK			SPHERIC	AL.	Bl	ADE	2000	RØDLIKE
8.33	3333		16.666	7		55.5	556	19.4444

RØCK	· ZING		BC/At2	1	JØL	
N UMBER	CLASS		SPHERICITY	. (CU FT	
1	IV		• 428819		3.35445	E-3
2	III		•358796	4	9.35715	E-3
3	III		• 332222		4.76685	E-3
4	III		• 30839		2.18922	E-3
5	II		•340278		2.82480	E-3
6 .	· III ·		•356653		4.23720	E-3
7 .	IV		.520833		4.30782	E-3
8	II		•288462		2.61294	E-3
9	IV		• 363636		2.64825	E-3
10	III		•292899		.003531	
11	II		• 388889		3.17790	E-3
12	III		.281065		3.70755	E-3
13	II		.28125		5.11995	E-3
14	III		• 304688		8.12130	E-3
15	1		.252551		3.35445	E-3
16	I		•143673		3.70755	E-3
17	III		. 425		2.25984	E-3
18	IV		.578512.		3.35445	E-3
19	III	all and	• 3		1.83612	E-3
20	III .		.328125		7.41510	E-3

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TABLE 8B (CONTINUED)

SHAPE CHARACTERISTICS 1 1/2 TO 3 INCH COBBLES

RØCK	 ZING		BC/At2	VØL	
NUMBER	CLASS		SPHERICITY	CU FT	
21	III		•340136	. 2.11860	E-3
22	II		.166205	5.64960	E-3
23	III		•34375	3.10728	E-3
24	III		• 3456	4.23720	E-3
25	IV		.546875	1.58895	E-3
26	III		• 443213	1.69488	E-3
27	IV	1.4	.660156	1.76550	E-3
28	III		• 388889	1.48302	E-3
29	III		•23625	9.18060	E-3
30	III		• 399584	8.82750	E-3
31	IV		. 428571	6.35580	E-3
32	II		.346939	4.59030	E-3
33	I		.21172	1.83612	E-3
34	III		. 446281	2.96604	E3
35	III		.278926	1.69488	E-3
, 36	III		•3375	1.58895	E-3

SPHERICITY CLASSIFICATION = [C*B]/ PERCENT--0.0 TØ 0.1 = 0PERCENT--0.1 TØ 0.2 = 5.55556 PERCENT -0.2 TØ 0.3 = 25 PERCENT -0.3 TØ 0.4 = 44.4444 PERCENT - 0.4 TØ 0.5 = 13.8889 PERCENT - 0.5 TØ 0.6 = 8.33333 PERCENT - 0.6 TØ 0.7 = 2.77778 PERCENT - 0.7 TØ 0.8 = 0 PERCENT. - 0.8 TØ 0.9 = 0 PERCENT - 0.9 TØ 1.0 = 0

TABLE 9A

SHAPE CHARACTERISTICS 3 TO 6 INCH CUBBLES

ZING CLASSIF	ICATIØNS:	PER CENT	PER CATEGORY	r		-
I DISK	II SPHERIC	AL	III BLADE		IV RØDLIKE	
12.5	12.5		56.25		18.75	
RØCK	ZING		BC/At2		VØL.	
NUMBER	CLASSIF	ICATION	SPHERICITY		CU FT	
1 .	III		.23358		.014124	
2	I		.242975		1.87143	E-2
3	III		.277345		3.14259	E-2
4	II		.243056		1.11227	E-2
5	· II ·		.194197		1.80081	E-2
6	VI		.478299		2.43639	E-2
7	III		.23615		7.76820	E-3
8	III		.192149		7.76820	E-3
9	III		•31901		1.87143	E-2
10	III	4	•23358		•014124	
11	· IV		.351727		2.45405	E-2
12	III		.346144		4.13127	E-2
13	III		.347107		1.13698	E-2
14	I		.206581		1.48302	E-2
15	III		.479917		8.82750	E-3
16	IV		.640523		•038841	1

SPHERICITY CLASSIFICATION = [C*B]/A:2 . PERCENT--0.0 TØ 0.1 = 0PERCENT - 0.1 TØ 0.2 = 12.5PERCENT -0.2 TØ 0.3 = 43.75PERCENT -0.3 TØ 0.4 = 25 PERCENT - 0.4 TØ 0.5 = 12.5 PERCENT - 0.5 TO 0.6 = 0 PERCENT - 0.6 TO 0.7 = 6.25 PERCENT - 0.7 TØ 0.8 = 0. PERCENT - 0.8 TØ 0.9 = 0 PERCENT - 0.9 TØ 1.0 = 0

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TABLE 9B

SHAPE CHARACTERISTICS 6 TO 9 INCH COBBLES

ZING CLASSIFICATIONS: PER CENT PER CATEGORY I II III IV DISK SPHERICAL BLADE RODLIKE 0 16.6667 44.4444 38.8889

RØCK	-14-24 C	ZING	BC/At2	VØL
N UMBER		CLASSIFICATIØN	SPHERICITY	CU FT
.1		III	•325939	.13559
2		IV	•332703	.149361
3		III ·	.524105	• 122879
4		IV	.617377	.13559
5		IV	.490484	8.40378 E-2
6		IV	•340909	.139475
7		III	•361531	•146183
8		IV	•657959	.121643
9	1.0	III	. 425381	8.56268 E-2
10		III	.368152	7.37979 E-2
11		III	•304311	•117935
12		III	.233091	9.63963 E-2
13		III	.356718	•117935
14		II	.265143	8.93343 E-2
15		IV	.72.4609	9.25122 E-2
16		II	• 1947	7.27386 E-2
17		IV	• 35114	7.98006 E-2
18		II ·	• 343158	.11052

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PHERICITY CLASSIFICATION = [C*B]/A12 PERCENT - -0.0 TO 0.1 = 0PERCENT--0.1 TØ 0.2 = 5.55556 PERCENT -0.2 TØ 0.3 = 11.1111 PERCENT -0.3 TØ 0.4 = 50PERCENT - 0.4 TØ 0.5 = 11.1111 PERCENT - 0.5 TØ 0.6 = 5.55556. PERCENT - 0.6 TO 0.7 = 11.1111 PERCENT - 0.7 TØ 0.8 = 5.55556 PERCENT - 0.8 TØ 0.9 = 0 PERCENT - 0.9 TØ 1.0 = 0

TABLE 10A SHAPE CHARACTERISTICS 1 1/2 TO 3 INCH ANGULAR

ZING	CLASSIFIC	ATIØNS:	PER	CENT	PER	CATEGØRY	
I		II		I	II	1.1	IV
DISK		SPHERIC	AL.	BI	ADE		RØDLIKE
28.2	2051	53.846	2		12.82	205	5.12821

	ZING		BC/At2	VØL	
	CLASS		SPHERICITY	CU FT	
	II	-	.261224	1.00629	E-2
	I		.265928	1.07337	E-2
	II		9.72222 E-2	1.07337	E-2
	Ι.		.145329 .	.004696	
	II		.272727	1.07337	E-2
-	II		.229601	1.34171	E-2
	II		.3025	8.05028	E-3
	I		.161157	7.37943	E-3
	II		• 33	1.27463	E-2
	I	,	.240741	1.00629	E-2
	II		•31856	8.72114	E-3
	III		.154959	8.72114	E-3
•	II		.176	•014038	
	II	4	•2112	. 1.47589	E-2
	IV		• 432099	8.72114	E-3
	III		.304688	8.72114	E-3
	II		.208333	1.34171	E-2
	II		.130178	1.54297	E-2
	II		•308642	1.00629	E-2
	TT		1 4605	1 00400	2-0
		ZING CLASS II I I I I I I I I I I I I I I I I I	ZING CLASS II I I I I I I I I I I I I I I I I I	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$

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TABLE 10B (CONTINUED)

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SHAPE CHARACTERISTICS 1 1/2 TO 3 INCH ANGULAR

I	RØCK	ZING	BC/At2	VØL
1	NUMBER	CLASS	SPHERICITY	CU FT
	21	VI	•5625	1.00629 E-2
	22	I	.11875	4.02514 E-3
	23	I	•1344	1.00629 E-2
	24	II	.206612	1.14046 E-2
	25	III	• 332 41	.009392
	26	I	•101977	•018784
	27	I	.202479	1.34171 E-2
	28	I	.285714	1.34171 E-2
	29	II	.356481	1.27463 E-2
	30	II	6.42792 E-2	1.14046 E-2
	31	II	•226843	1.34171 E-2
	32	I	•153846	1.47589 E-2
	33	II	•213843	1.54297 E-2
	34	TTT	•216049	6.70857 E-3
'	35	TT ·	8.73016 E-2	6.70857 E-3
	36	· ī	.176871	7.37943 E-3
	37	TT	.333333	1.00629 E-2
	38	TTT	2.3	.009392
	39	TT	.2625	.009392
		* *		-00/0/2

n mir		227	01 01				1.00	F	11.10
SPH	TRICI	11	ULA:	2274	TCA	111	11V =	rc*B1	141S
PER	CENT	-	0.0	TØ	0.1	=	7.6	9231	
PER	CENT	-	0.1	TØ	0.2	=	28.	2051	
PER	CENT	-	0.2	ΤØ	0.3	=	38.	4615	
PER	CENT	-	0.3	ΤØ	0.4	=	20.	5128	
PER	CENT	-	0.4	ΤØ	0.5	=	2.5	641	
PER	CENT	-	0.5	TØ	0.6	=	2.5	641	
PER	CENT	-	0.6	TØ	0.7	==	0		
PER	CENT	-	0.7	ΤØ	0.8	=	0		
PER	CENT	-	0.8	TØ	0.9	=	0	37.	
PER	CENT	-	0.9	TØ	1.0	=	0		- are

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TABLE 11A

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SHAPE CHARACTERISTICS
3 TO 6 INCH ANGULARZING CLASSIFICATIONS: PER CENT PER CATEGORYIIIIIIIIIVDISKSPHERICAL13.888938.88892522.2222

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RØCK	ZING	BC/At2	VØL	
NUMBER	CLASS	SPHERICITY	CU FT	
1	III	•293878	3.46038 E-	2
, 2	III	34	8.82750 E-	3
3	II	• 325255	3.10728 E-	2
4	II	• • 160494	4.09596 E-	2
5	II	•376033	.010593	
6	I	•168006	3.03666 E-	2
7	III	• 315	.03531	
. 8	II	• 344388	2.68356 E-	2
9	I	• 163265	1.12992 E-	ż
10	IV	.52	1.90674 E-	2
. 11	IV	•7	2.33046 E-	2
12	III	• 3875	.007062	
13	II	•227556	·4.62561 E-	2
14	III	•555269	1.97736 E-	2
15	II	.250488	.028248	
16	II	•341644	2.64825 E-	2
17	III	•23638	2.01267 E-	2
18	II	•243896	2.96604 E-	2
19	III	• 3894	1.48302 E-	2
20	III	• 312757	2.57763 E-	2

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TABLE 11B(CONTINUED)

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SHAPE CHARACTERISTICS 3 TO 6 INCH ANGULAR

	RØCK	ZING	BC/At2	VØL	
	NUMBER	CLASS	SPHERICITY	CU FT	
	21	I	•152778	2.54232 E-2	
	22	II	• 318611	3.38976 E-2	
	23	. IV	•534343	1.97736 E-2	
	24	II	.247934	1.55364 E-2	
	25	IV	•580499	1.87143 E-2	
-	- 26	III	• 339923	2.93073 E-2	
	27	IV	• 412722	2.33046 E-2	
	28	II	• 432825	1.12992 E-2	
	29	IV	• 410156	2.29515 E-2	
	30	II	•311111	3.24852 E-2	
	31	I	• 181474	1.23585 E-2	
1	.32	II	•286458	1.69488 E-2	
	33	IV	• 486111	2.57763 E-2	
	34	IV	• 453686	3.10728 E-2	
	35	r	.15576	6.35580 E-3	
	36	II	•23615	. 3.60162 E-2	

SPHERICITY CLASSIFICATION = [C*B]/A 2

P	ER	CE	N	T		0	• 0) 1	rø	0	•	1	=	0							
P	ER	CE	N	T		0	• 1	1	rØ	0		2	=	1	6.	6	6	6.7		•	
P	ER	CE	N	T	-	0	.2		ΓØ	0		3	=	2	2.	2	2	22			
P	ER	CE	N	T	-	0	• 3	3 7	r.Ø	0		4	=	3	3	. 3	3	33	;		
P	ER	CE	N	Т	-	1	0.	4	T	Ø	0	• 5	5	==	1	13	•	88	8	9	
P	ER	CE	N	T	-	1	0.	5	T	Ø	0	• 6	5	==		11	•	11	1	1	
P	ER	CE	N	T	-	1	0.	6	Т	Ø	0	• 7	7	==	2	2.	7	77	7	8	
0	ER	CE	N	Т	-	1	0.	7	T	Ø	0	• 8	3	=	()					
P	ER	CE	N	T	-	1	0.	8	T	Ø	0	• 5)		()					
D	ER	CE	N	T	-	1	0.	9	T	Ø	1	• ()		()					

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TABLE 12A SHAPE CHARACTERISTICS 6 TO 9 INCH ANGULAR

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ZING	CLASSIF	ICATIØNS:	PER	CENT	PER	CATEGØRY	
I		II		I	II		IV
DISK		SPHERIC	AL	BL	ADE		RØDLIKE
5.55	5556	27.777	8	1	13.88	389	52.7778
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RØCK	ZING	BC/At2	VØL
N UMBER	CLASS	SPHERICITY	CU FT
1	II	•3364 .	.279077
2	IV	• 408163	•13283
3	IV	. 4292	.24285
4	IV	• 350911	.179119
5	I	.268647	.181131
6	II	• 306777	.191865
7	III	.505003	.191865 .
8	IV	•510381	.138197
9	IV	•792549	.214674
10	IV	• 395703	•15631
11	· IV ·	•512397	.193207
12 .	IV	• 402883	.165702
13 .	III	•340741	.186498
14	IV	• 453616	.118742
15	II	.214249	.145576
. 16	II	•274921	•314632
17	II	• 327168	.243521
18	IV	•763158	.231446
19	II	.279952	.116729
20	IV	• 442151	.17241

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1997 - 1997 - 1997 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 - 1997 -

TABLE 12B (CONTINUED)

SHAPE CHARACTERISTICS 6 TO 9 INCH ANGULAR

	R ØCK	ZING	BC/At2	VØL
	NUMBER	CLASS	SPHERICITY	CU FT
	21 .	IV	• 446281	.268343
	22	IV	•622222	.235471
	23	IV	• 385614	•130817
1	24	IV	•531636	.175094
1	25	III	.233918	.17979
'	26	II	• 444444	.157651
	27	IV	•713668	.12.4779
_	- 28	II	•31238	•193878
	29	III	•575	•160335
	30	IV	•76	•188511
	31	II	•232861	.188511
	32	IV	•634039	•149601
	33	III	• 400549	129475
-	34	II	•277686	.189853
	35	IV	•5775	• 122096
	36	1	•186057	.173081

SPHERICITY CLASSIFICATION = [C*B]/At2

FCK	CEN1		0.0	10	0.1		0	
PER	CENT	-	0.1	ΤØ	0.2	=	2.77778	
PER	CENT		0.2	TO	0.3	=	19.4444	
PER	CENT		0.3	TØ	0.4	==	22.2222	
PER	CENT	-	0.4	ΤØ	0.5	=	22.2222	
PER	CENT		0.5	ΤØ	0.6	=	16.6667	
PER	CENT	~	0.6	TØ	0.7	=	5.55556 .	
PER	CENT	••	0.7	TØ	0.8	=	11.1111	
PER	CENT	-	0.8	TØ	0.9	=	0	
PER	CENT		0.9	TØ	1.0	=	0	

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VELOCITIES AT SIGNIFICANT POINTS Cobbles

TEST	Size Category	Naximum Boundary Velocity (fps)	Distance from Datum (ft)	Velocity at d(in) from rock surface	d from Datum (ft)	Velocity at • (fps)	Distance from Datur (ft)	Hean Boundary Velocity (fos)
	22 cfs							
11 11	a 3 "=d=12" b 3 "=d=12"	6.4 7.1	1.48 0.78	5.1 6.4	0.0625 0.125	5.6 6.7	0.28 0.19	5.8 6.3
10 10	22 cfs a 1½"≤d≤3" b 1½"≤d≤3"	7.3 8.2	1.00 .89	6.5 7.5	0.125 0.25	6.7 7.4	0.18 02	6.0 6.4
55	21 cfs a 3"≤d≤6" b 3"≤d≤6"	8.1 9.9	.92 .43	7.7 9.9	0.25 0.5	7.6 8.8	0.18 - 0.02	6.5 7.1
0, 00	20.6 cfs a ó"≤d≤9" b ó"≤d≤9"	9.) ₁ 10.8	.67 .26	9.4 7.0	0.5 0.75	8.8 10.7	0.00	·7•1 7•4
1	1 <u>h</u> cfs a 3"≤d≤6"	9.0	•43	8.5	0.25	8.4	0.18	6.3
777	14 cfs a 6"≤d≤9" b 6"≤d≤9"	9.0 10.0	•20 •33	6.4 10.0	0.5 0.75	9.0 10.0	0.12 0.23	6.0 6.0
66	11.6 cfs a 6"ede9" b 6"ede9"	8.4 9.7	.1♀ .19	11.5 4.7	°0.5 0.75	8.4 9.7	0.19 0.19	5.1 5.7
1	7.5 cfs a 3"sds6"	ő . 8	.18	6.6	0.25	6.8	0.18	4.3
2	7.5 cfs b 3"≤d<6"	8.0	10	5.0	0.5	8.0	10	0:0
3	7.5 cfs a 3"≤d≤6" b 3"≤d≤6"	8.2 7.5	0.00 18	5.3 7.5	0.25 0.5	8.2 7.5	0:00 18	6.0 6.0

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VELOCITIES AT SIGNIFICANT POINTS Angular

$\begin{array}{c ccccccccccccccccccccccccccccccccccc$			with the second s						
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	TEST NØ•	T SIZE • CATEGØRY	MAXIMUM JET VELOCITY [FPS]	DISTANCE FRØM DATUM EFTJ	VELØCITY AT MAX RØCK DIA FRØM DAT LINJ	DISTANCE FRØM DATUM EFTJ	VELØCITY AT SIGN [FPS]	DISTANCE FRØM BASE ØF.VEL PRØFILE LFTJ	「「「「「「「」」」」」」」」」」」」」」」」」」」」」」」」」」」」」」」
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	18 19 18 19 12 13 14 15 16 17 12 13 14 15 16	1.5 1.5 3 3 3 3 3 3 3 3 3 3 3 3 3 6 6 6 6 6	9.2 8.2 10.2 11.1 8.6 8.4 8.9 11.2 8.4 8.4 8.4 8.4 8.4 10.3 10.4 10.5 10.6 8.5	1.08 1.23 .64 .82 .9 1.07 1.2 .7 .6 .49 .52 .7 .4 .7 .37	8 • 1 4 • 7 5 • 8 8 • 5 8 • 3 7 • 9 8 • 5 11 8 • 2 8 • 8 9 6 • 8 9 • 9 9 • 1 6 • 9	.25 .25 0 .5 .5 .5 .5 .5 .5 0 0 0 0	8 7.5 8.7 9.5 8.2 8.2 8.1 10.5 7.1 8 9 9.1 9.9 9.1 6.9	• 35 • 46 • 21 • 36 • 37 • 35 • 41 • 41 • 41 • 2 • 34 • 3 • 25 • 2 • 25 • 15	
21 9 11.4 .05 11.2 0 0	17 20 21 20 21	6 6 9	9.7 10 10.2 12.9	•39 •02 •62 •23	4.7 8.7 9.3 10.5	0 •75 •75 0	7 •2 10 .8 •1 12 • 3	•2 •52 •35 •4	

TABLE 15A 3/4 TO 1 1/2 INCH COBBLES

PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=DIAMA PIE 1 IS RATIØ	ETER ØF SPHERE Ø ØF MEAN AXIS DI	OF WEIGHT EQUAL	TØ RØCK SAMPLE WT SPHERE	
PIE 2 IS RATIØ	OF MEAN AXIS DI	M AND SIGNIFICA	NT DEPTH	
PIE 3 IS RATIØ	ØF RØCK VØLUME	AND ABC RECTANG	ULAR SØLID	1
RØCK	DIA	PIE 1	PIE 2	PIE 3
N UMBER	INCHES			
1 .	1.69442	1.00821	6.10119	•598937
2	.1.51045	1.07584	5.80357	.525042
3	1.4158	1.05947	5.35714	.667522
4	1.48023	1.0978	5.80357	.560718
5	•932532	1.07235	3.57143	.585751
6	1.48023	1.01335	5.35714	.610157
7	1.51045	1.02067	5.50595	•556863
8	1.59452	•992984	5.65476	56199
9	1.4158	1.05947	5.35714	•510458
10	1.22193 .	1.05707	4.6131	• 433889
11	1.17489	1.02847	4.31548	• 497
12	1.30645 .	1.11626	5.20833	• 429121
13 .	•847269	1.08191	3.27381	• 3905
14 .	1.38126	1.02564	5.05952	.660847
15	•932532	1.02767	3.42262	•506204
16	1.00453	1.07845	3.86905	.507143
17	1.12375	1.07527	4.31548	.267772
18	1.56749	•956942	5.35714	•540693
19	1.51045	1.13101	6.10119	• 450577
20	1.5395	.974343	5.35714	.62759

TABLE 15B 3/4 TO 1 1/2 INCH COBBLES

CONTINUED

PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=D	DIAMETER OF SPHERE	OF WEIGHT EQUAL	TØ RØCK SAMPLE	
PIE 1 IS RA	TIØ ØF MEAN AXIS D	IM AND DIA ØF =	WT SPHERE	
PIE 2 IS RA	TIØ ØF MEAN AXIS D	IM AND SIGNIFIC	ANT DEPTH	
PIE 3 IS RA	TIØ ØF RØCK VØLUME	AND ABC RECTAN	GULAR SØLID	
RØCK	DIA	PIE 1	PIE 2	PIE 3
NUMBER	INCHES			
21	1.12375	1.00112	4.01786	• 488125
22	1.5395	1.00141	5.50595	.535981
23	1.2656	1.05352	4.7619	.694223
24	1.64599	1.01256	5.95238	.608187
25	1.22193	.988874	4.31548	.284
26	1.12375	1.03819	4.16667	• 447 448
27	1.51045	1.02067	5.50595	.670556
28	1.51045	1.04825	5.65476	• 488125
29	1.38126	1.0558	5.20833	•557858
30 .	1.48023	1.01335	5.35714	• 520667
31	1.30645	1.14815	5.35714	• 493489
32	1.5395	1.10967	6.10119	•619842
33	1.22193	1.05707	4.6131	.5325
34	1.56749	1.03669	5.80357	• 597236
35	1.48023	1.12595	5.95238	• 426277
36	1.71763	1.01884	6.25	•548071
MEAN DIA ØF	= WEIGHT SPHERE 1	.36891 INCHES		

T	AB	LE	14	۶F	1
*	· • [_]	frank laund		• •	•

L 1/2 TO 3 INCH COBBLES PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

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WHERE DIA:	DIAMETER ØF SP	HERE ØF WEIGHT EQU	JAL TØ RØCK SAM	PLE
PIE 1 IS H	RATIØ ØF MEAN A	XIS DIM AND DIA ØF	= WT SPHERE	100 T
PIE 2 IS I	RATIØ ØF MEAN A	XIS DIM AND SIGNIF	ICANT DEPTH	
PIE 3 IS H	RATIØ ØF RØCK V	ØLUME AND ABC RECT	ANGULAR SØLID	
RØCK	DIA	PIE 1	PIE 2	PIE 3
NUMBER	INCHES			
1	2.25297	1.03567	11.6667	.500641
2	3.17672	1.07553	17.0833	• 49 45 4
3	2.5479	1.07932	13.75	• 470168
4	1.38889	1.01471	9.58333	.67818
5	2.04698	1.05847	10.8333	.531293
6	. 2.39175	1.04526	12.5	.534018
7	2.41531	. 1.01781	12.2917	. 529345
8	2.03061	. 1.10804	11.25	• 455969
9	2.0631	•989612	10.2083	.605114
10	2.25297	1.05416	11.875	. 606838
11	2.1549	1.04413	11.25	.522992
12	2.2925	• 9996.35	11.4583	.664009
13	2.57898	1.06631	. 13.75	• 491515
14	2.88013	1.01269	14.5833	•719672
15	2.22583	1.06702	11.875	.535318
16	2.2925	1.18139	13.5417	.5325
17	1.88889	1.03677	9.79167	•588048
18	2.19801	1.06157	11.6667	• 481786
19	1.78823	1.00192	8.95833	•676867
.20	2.8385	1.02754	14.5833	•610157

TABLE 168 (CONTINUED) 1 1/2 TO 3 INCH COBBLES PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=DIAM	ETER ØF SPHERE	OF WEIGHT EQUAL	. TØ RØCK SAMF	PLE	
PIE 1 IS RATIØ	ØF MEAN AXIS	DIM AND DIA ØF =	WT SPHERE		
PIE 2 IS RATIØ	ØF MEAN AXIS	DIM AND SIGNIFIC	CANT DEPTH		1.2
PIE 3 IS RATIØ	OF ROCK VOLUM	IE AND ABC RECTAN	GULAR SØLID		
RØCK	DIA	PIE 1	PIE 2		PIE 3
NUMBER	INCHES				
21	1.90777	1.00466	9.58333		.595048
22	2.6487	1.08544	14.375		.548071
23	2.14014	1.0708	11.4583		.578519
24	2.37979	962969	11.4583		.694223
25	1.69867	.981159	8.33333		.62759
26	1.7446	1.07475	9.375		• 493264
27	1.7446 .	1.0031	8.75		.577663
28	1.67475	. 1.02006	8.54167		.578519
29	3.12182	1.0811	16.875		.537196
30 .	3.05037	•969829	14.7917	1	.656083
31	2.76925	•978002	13.5417		.597705
32	2.50526	1.01453	12.7083		.533246
33	1.76668	1.06131	9.375	18.1	.630622
34	2.14014	1.01239	10.8333		.552223
35	1.7446	1.09863	9.58333		.504889
36	1.69867	1.07927	9.16667		.520667
MEAN DIA ØF = 1	WEIGHT SPHERE	2.24837 INCHES			

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TABLE 17R

3 TO 6 INCH COBBLES

PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

DIA =	DIAMETER	ØF SPHERE ØF	WT = RØCK SAME	PLE			
PIE 1	IS RATIØ	OF MEAN AXIS	DIM AND DIA ØF	F = WT SPHERE			
PIE 2	IS RATIØ	ØF MEAN AXIS	DIM AND SIGNIA	FICANT DEPTH			
PIE 3	IS RATIØ	ØF RØCK VØLUN	ME AND ABC RECT	TANGULAR SØLID		1	
RØCK		DIA	PIE 1	PIE 2		PIE 3	
N UMBER	2	INCHES					
1		3.58495	1.15064	31.7308		.587081	
2		3.92567	. 1.17814	35.5769		• 409579	1
3		4.79906	1.11133	41.0256		• 441664	
4		3.65798	1.0935	30.7692		•366094	
5		4.43254	1.04342	35.5769	- 4	.399469	
6		4.42412	1.08308	36.859		• 407 509	
7		3.17168	1.05097	25.641		.530391	
8		3.21572	1.12728	27.8846		• 419893	
9		4.2172 .	1.02754	33.3333		• 469309	
10		3.58495	1.15064	31.7308		• 587081	
11		4.53439	1.07512	37.5		• 41 46 33	
12		5.30032	1.06126	43.2692		• 465213	
13		3.72105	1:08616	31.0897		•340209	
14		3.49158	1.14561	30.7692		•539862	
15		3.60844	1.06232	29.4872		•296575	
16		5.24377	1.06476	42.9487		• 40 4 4 4 4	
MFANI	DTA OF = 1	JT. SPHERE IS.	4.05709				

 $4EAN DIA OF = WI \cdot SPHERE IS 4 \cdot 0570$

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	TABLE 178	
	6 TO 9 INCH COBBI	ES
	PIE TERMS FØR PHYSICAL CHARAC	TERISTICS ØF RIPRAP
DIA = DIAMETER	R ØF SPHERE ØF WT = RØCK SAMPLE	
PIE 1 IS RATIO	Ø ØF MEAN AXIS DIM AND DIA ØF =	WT SPHERE
PIE 2 IS RATIO	Ø ØF MEAN AXIS DIM AND SIGNIFIC.	ANT DEPTH
PIE 3 IS RATIO	3 OF ROCK VOLUME AND ABC RECTAN	GULAR SØLID
RØCK	DIA PIE 1	PIE 2 PIE 3
N UMBER	INCHES	and the second
1	7.72019 1.13339	43.75 .443121
2	7.93068 1.05077	41.6667 .510072
3	7 • 48214 1 • 03023	38.5417 .51189
4	7.79417 1.01572	39.5833 .499487
5	6.44699 1.05993	34.1667 .482099
6	7.7876 . 1.02727	40 .531156
7	7.76232 1.1004	42.7083 .459412
8	7.41144 1.03444	38.3333 .476878
9	6.64997 1.11529	37.0833 .405699
10	6.31812 1.11452	35.2083 .437659
11	7 • 32578 1 • 12047	41.0417 .455005
12	6.80988 1.15029	39.1667 .454884
13	7.31338 1.12807	41.25 .444196
14	• 6.71544 1.09201	36.6667 .468658
15	6.78688 1.0621	36.0417 .430892
16	6.28802 1.0301	33.9583 .519658
17	6.47405 1.17778	38.125 .351646
18	6.60458 1.17974	38.9583 .447987
MEAN DIA ØF =	WT. SPHERE IS 7.09009	

TABLE 18 R 1 1/2 TO 3 INCH ANGULAR PIE TERMS FOR PHYSICAL CHARACTERISTICS OF RIPRAP

WHERE DIA=DIAMETER ØF SPHERE ØF WEIGHT EQUAL TØ RØCK SAMPLE PIE 1 IS RATIØ ØF MEAN AXIS DÍM AND DIA ØF = WT SPHERE PIE 2 IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID

RØCK	DIĄ	PIE 1	PIE 2	PIE 3
NUMBER	INCHES			
1	2.69417	1.09805	7.39583	•794908
2	2.85258	1.13932	8.125	.650802
3	2.77907	1.16946	8.125	883231
4	2.0477	1.28193	6.5625	• • 727367
5	3.01274	1.24472	9.375	• 408768
6	3.13606	1.24891	9.79167	.467 495
7	2.79951	1.25022	8.75	•367891
8.	2.51455	1.35876	8.54167	• 475586
9	3.04737	1.17588	8.95833	•533953
10	2.8784	1.05672	7.60417	•792643
11	2.73019	1.23618	8.4375	• 441 413
12	2.7159	. 1.28871	8.75	• 584538
13	3.21488	1.19237	9.58333	• 566553
14	3.29001	1.2158	10	• 49 46 09
15	2.72306	1.28532	8.75	• 332734
16	2.42778	1.20137	7.29167	•772827
17	3.17858	1.20599	9.58333	.515218
18	3.19424	1.17399	9.375	•745805
19	3.13606	1.00976	7.91667	•618262
20	2.33291	1.04427	7.39583	•951172

TABLE 188 (CONTINUED) 1 1/2 TO 3 INCH ANGULAR PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=DIAMETER ØF SPHERE ØF WEIGHT EQUAL TØ RØCK SAMPLE PIE 1 IS RATIØ ØF MEAN AXIS DIM AND DIA ØF = WT SPHERE PIE 2 IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID

ł	R ØCK N UMBER	DIA INCHES	PIE 1	PIE 2	PIE 3
	21	2.74433	1.21462	8.33333	• 483017
	22	2.0477	1 • 40 40 1	7.1875	• 468577
	23	2.82629	1.32683	9.375	•529939
	24	3.47081	1.00841	8.75	.573297
	25	2.77907	1.25941	8.75	• 455561
	26	3.17333	. 1.36555	10.8333	.683792
	27	3.03015	1.18256	8.95833	•688232
	28	3.09783	1.18361	9.16667	.560782
	29	3.05874	1.07615	8.22917	.678036
	30	2.91627	1.42876	10.4167	.545997
	31	3.06439	1.22374	9.375	•537619
	32	3.20974	1.22024	9.79167	.603628
	33	3.24535	1.09131	8.85417	•749408
	34	2.56386 .	1.20261	7.70833	.588821
	35	2.48915	1.12154	6.97917	•917643
	36	2.60355	1.2803	8.33333	• 498233
	37	2.83291	1.14723	8.125	.572465
	38	2.76528	1.26569	8.75	• 463696
	39	2.73019	1.23618	8.4375	• 49 4609
	MEAN DIA ØF =	WEIGHT SPHERE	2.86807 INCHES		

TABLE 19A

TO 6 INCH ANGULAR PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

W RE DI A=DIAMETER ØF SPHERE OF WEIGHT EQUAL TO ROCK SAMPLE PIE MEAN AXIS DIM AND DIA OF = WT SPHERE IS RATIØ 1 ØF IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH PIE 2 PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID

RØCK	DIA	PIE 1	PIE 2	PIE 3
NOWREK	INCHES			
1	4.36704	1.41209	7.90598	• 303723
5	2.81333	1.33294	4.80769	• 358916
3	4.07952	1.22564	6.41026	• 481289
4	4.53355	1.1948	6.94444	.604942
5	2.86449	1.4255	5.23504	•292583
6	. 4.12309	1.4148	7.47863	• 39463
7	4.31847	1.24465	6.89103	• 459142
8	3.96922	. 1.28069	. 6.51709	• 392566
9	3.05774	1.41717	5.55556	•348661
10	3.56624	1 • 1 45	5.23504	•506899
1.1	3.80555	1.17153	5.71581	• 460233
12	2.71749	· 1.39528	4.86111	•251936
13	4.80055	1.27589	. 7.85256	• 42.6296
14	3.80555	1.22628	5.98291	• 369861
15	4.11769	1.30534	6.89103	•380605
16	3.95755	1.30552	6.62393	• 370318
17	3.66777	1.37459	6.46368	• 406771
18	4.09596	1.30209	6.83761	• 410436
19	3 • 11544	1.32405	5.28846	• 423799
20	3.94582	1.22493	6.19658	• 463069

TABLE 19B (CONTINUED) 3. TO 6 INCH ANGULAR PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=DIAMETER ØF SPHERE ØF WEIGHT EQUAL TØ RØCK SAMPLE PIE 1 IS RATIØ ØF MEAN AXIS DIM AND DIA ØF = WT SPHERE PIE 2 IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID

RØCK	DIA	PIE 1	PIE 2	PIE 3
N UMBER	INCHES		2	Anne Carlo de
21	4.10142	1.32068	6.94444	.394445
22	4.30368	1.23925	6.83761	• 435781
23	. 3.67796	1.21218	5.71581	•.411788
24	. 3.36409	1.08994	4.70085	•650834
25	3.43543	. 1.28562 .	5.66239	• 38498
26	4.02936	1.27191	6.57051	. 434354
27	3.71816	1.33355	6.35684	.355294
28	2.95642	1.24024	4.70085	. 420918
29	3.76074	· 1.21873	5.87607	• 447 66 4
30	4.26122	1.2516	. 6.83761	. 427691
31	2.92463	1.22523	4.59402	.619
32	3.43543	1.21285	5.34188	• 473334
33	3.47001	1.39289	6.19658	. 42 420 4
34	4.17891	1.07683	5.76923	.622537
35	2.34657	1.40276	4.22009	• 45 407
36	4.48853	1.40172	8.06624	• 307386
MEAN DIA	OF - WEIGHT SPHERE	3.79707. TNCH	FS	

EAN DIA ØF = WEIGHT SPHERE 3.72707 INCHES

TABLE 209 6 TO 9 INCH ANGULAR PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=DIAMETER ØF SPHERE ØF WEIGHT EOUAL TØ RØCK SAMPLE PIE 1 IS RATIØ ØF MEAN AXIS DIM AND DIA ØF = WT SPHERE PIE 2 IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID

RØCK NUMBER	DIA	PIE 1	PIE 2	PIE 3
1	8.60801	1.04554	18	.733975
2	6.76744	.985109	13.3333	•839423
3	7.75684	1.24621	19.3333	• 500602
4	7.41641	1.18543	17.5833	• 510439
5	7.43663	1.17661	17.5	.562115
6	7.53239	1.28335	19.3333	• 415728
7	7.55013 .	1.18099	17.8333	•52847
8	6.78133	. 1.0261	13.9167	.761886
9	7.93541	1.08165	17.1667	• 591388
10	7.09779 .	. 1.10363	15.6667	.611217
11	7.52208 .	1.04138	15.6667	•730731
12	7.29769	. 1.1933	17.4167	• 467 30 3
13	7.5863	1.09298	16.5833	•664258
14	6.41546	1.13008	14.5	.571523
15	6.8882	1.25819	17.3333	• 477214
16	8.96762	1.12442	20.1667	.616258
17	8.27491	1.21351	20.0833	• 468732
18	8.05259	1.08661	17.5	.611234
19	6.40641	1.02762	13.1667	•840368
20	7.36782	1.08553	15.996	.625698

N

TABLE 20B (CONTINUED) 6 TO 9 INCH ANGULAR PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP

WHERE DIA=DIAMETER ØF SPHERE ØF WEIGHT EQUAL TØ RØCK SAMPLE PIE 1 IS RATIØ ØF MEAN AXIS DIM AND DIA ØF = WT SPHERE PIE 2 IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID

R ØCK				DIA		PIE 1	PIE 2		PIE 3
NUMBE	R		۰.	INCHES					
21				8.59077		.999134	17.1667		•780634
22				8.1465		1.1866	19.3333		.45928
23				6.69239		1.20784	16.1667		• 47 1879
24				7.3344		. 1.00554	14.75		.780679
25				7.41931 .		1.30291	19.3333		. 458987
26				7.0639	•	1.15611	16.3333		.529488
27				6:55314	. '	.1.16992	15.3333	1 a.	• 491964
28				7.62668		1.18553	18.0833		•517438
29				7.18321	•	1.19492	17.1667		• 481841
30				7.55107		1.21396	18.3333		. 428614
31				7.57521		1.47411	22.3333		.284732
32				. 6.99945		1.14295	16		.515154
33		•		6.69239		1.18294	15.8333		.538134
34				. 7.53613	1.	1.26059	. 19		.454463
35				6.59859		1.27553	16.8333		•365337
36				7.31063		1.22539	17.9167		.555523
MEAN	DIA	ØF	=	WEIGHT SPHE	RE	7.40376 INCHE	S		

TEST	vmax	h _B	D	dave	[™] ² /h _B g	he/D	02/09	v%g
	(FPS)	(FT)	(FT)	(IN)	II-4	II - 5	II-6	I I- 7
· · · · · · · · · · · · · · · · · · ·			COBBI	ъs				
loD								
1 1/2" TO 3"	8.63	1.01	1.76	2.25	2.29	•57	1.31	1.03
11D 3/4" TO 1 1/2"	7.00	•54	2.18	1.37	2.84	•25	•70	1.11
	ž.		ANGULAR	RIPRAP			a.	
19B 1 1/2" TO 3"	11,10	1.18	1.88	2.87	3.24	.63	2.04	1.33
14B 3" TO 6"	10,50	2.14	1.46	3.73	1.60	1.47	2.35	•92
15B 3" TO 6"	10.60	1.59	1.07	3.73	2.20	1.49	3.28	•94
20B 6" TO 9"	12.90	2.42	•79	7.40	2.14	3.05	6.53	•70

PI-TERMS FOR VELOCITY PARAMETERS

DISCHARGE

WHERE

G = FLØW IN CUBIC FEET PER SECØND IN GRAVEL 0 = TØTAL DISCHARGE FØR UNIT WIDTH SECTIØN F = DISCHARGE IN JET CØMPUTED BY THE CØNTINUITY EQUATIØN E = DISCHARGE IN JET CØMPUTED BY EQUATIØN 3.16

T.EST N UMBER		DISCHARGE IN DIDDAD		TØTAL DISCHARGE	DISCHARGE F		DISCHARGE E	
		G	۰.	FLUME				
		[CFS/FT]		CCFS/FTJ	[CFS/FT]		CCES/FT3	
10	• ,	3.94101	1	10.925	6.98399		10.3367	
14		4.54291		11.15	6.60709		15.0458	
15 .		1.61315		1.1	9.38685		8.86255	
19.	-	-3.01951		10.735	13.7545		18.7739	
20		5.97264		10.91	4.93736	1.22	. 6.55503	
	T.EST NUMBER 10 11 14 15 19 20	T.EST N UMBER 10 11 14 15 19 20	T.EST DISCHARGE NUMBER IN RIPRAP G CCFS/FTJ 10 3.94101 11 10.6949 14 4.54291 15 1.61315 19 -3.01951 20 5.97264	T.EST DISCHARGE NUMBER IN RIPRAP G ICFS/FTJ 10 3.94101 11 10.6949 14 4.54291 15 1.61315 19 -3.01951 20 5.97264	T.EST DISCHARGE TØTAL NUMBER IN DISCHARGE RIPRAP IN G FLUME ICFS/FTJ ICFS/FTJ 10 3.94101 10.6949 11.045 14 4.54291 15 1.61315 19 -3.01951 20 5.97264	T.EST DISCHARGE TØTAL DISCHARGE F NUMBER IN DISCHARGE F RIPRAP IN FLUME FLUME ICFS/FTJ ICFS/FTJ ICFS/FTJ ICFS/FTJ 10 3.94101 10.925 6.98399 11 10.6949 11.045 .350077 14 4.54291 11.15 6.60709 15 1.61315 11 9.38685 19 -3.01951 10.735 13.7545 20 5.97264 10.91 4.93736	T.EST DISCHARGE TØTAL DISCHARGE NUMBER IN DISCHARGE F RIPRAP IN FLUME ECFS/FTJ ECFS/FTJ 10 3.94101 10.925 6.98399 11 10.6949 11.045 .350077 14 4.54291 11.15 6.60709 15 1.61315 11 9.38685 19 -3.01951 10.735 13.7545 20 5.97264 10.91 4.93736	T.EST DISCHARGE TØTAL DISCHARGE <t< td=""></t<>

COMPARISON OF VELOCITY COMPUTATION

WHERE V = MEAN VELØCITY ØF JET BY VELØCITY PRØFILE AVERAGING U = MEAN JET VELØCITY BY EQUATION 3.15

L = MEAN DEPTH OF JET ABOVE RIPRAP SURFACE

TEST	V	U	L
NUMBER	[FPS]	CFPSJ	
10	6.71527	12,9705	1.04
10	5.83462	. 11.5432	• 06
14	 4.93067	13.4744	1.34
15	8.49489	12.3163	1.105
19	7.33574	12,0363	.1.875
20	7.53795	14.747.5.	.655



STABILITY OF RIPRAP HYDRAULIC FLUME STUDY, 1:1 SCALE FIGURE 1



Figure 2 General view of test flume with baffle platform and test section in the center of the picture Photo PX-D-61604NA



Figure 3 Dye test showing jet under the baffle and differential pressure recording equipment Photo PX-D-61605NA



Figure 4B 3- to 6-inch angulars 6-inch scour in progress Photo PX-D-61501NA

80



Figure 5A Test 8 Before and after 6-inch scour 3- to 6-inch cobbles 1-foot stone layer 21-cubic-foot-per-second flow Photos PX-D-61607NA and PX-D-61608NA

Figure 5B Test 6 Before and after 9-inch scour 6- to 9-inch cobbles 1-foot stone layer 11.6-cubic-foot-per-second flow Photos PX-D-61609NA and PX-D-61610NA







Figure 6A Test 7 Before and after 9-inch scour 6- to 9-inch cobbles 1-foot stone layer 14-cubic-foot-per-second flow Photos PX-D-61611NA and PX-D-61612NA Figure 6B Test 8 Before and after 9-inch scour 6- to 9-inch cobbles 1-foot stone layer 20.6-cubic-foot-per-second flow Photos PX-D-61615NA and PX-D-61614NA







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Figure 7A Test 9 Before and after 9-inch scour 6- to 9-inch cobbles 1-foot stone layer 7.5-cubic-foot-per-second flow Photos PX-D-61615NA and PX-D-61616NA Figure 7B Test 10 Before and after 3-inch scour 1-1/2- to 3-inch cobbles 1-foot stone layer 22-cubic-foot-per-second flow Photos PX-D-61617NA and PX-D-61618NA





Figure 8B Test 12 Before and after 6-inch scour 3- to 6-inch angulars 1-foot stone layer 22-cubic-foot-per-second flow Photos PX-D-61621NA and PX-D-61622NA



Figure 8A Test 11 Before and after 1-1/2-inch scour 3/4- to 1-1/2-inch cobbles 1-foot stone layer 22-cubic-foot-per-second flow Photos PX-D-61619NA and PX-D-61620NA



Figure 9A Test 18 Before and after 3-inch scour 1-1/2- to 3-inch angulars 1-1/2-foot stone layer 22-cubic-foot-per-second flow Photos PX-D-61623NA and PX-D-61624NA Figure 9B Test 20 Before and after 9-inch scour 6- to 9-inch angulars 1-1/2-foot stone layer 22-cubic-foot-per-second flow Photos PX-D-61625NA and PX-D-61626NA





FIGURE 10

SINGLE CHANNEL ELECTRONIC RECORDER DATA TAPE



.86



FIGURE 11

SINGLE CHANNEL ELECTRONIC RECORDER DATA TAPE



FIGURE 12













94 VELOCITY PROFILES 105 TEST 100 Q= 22.0CFS INITIAL INSTABILITY 1.7 3" SCOUR TESTS 1.6 STREAM ROUNDED RIPRAP ۵ 1.5 V JSW 3 APR. 68 Ø 1.4 V TEST 10c Q= 21.95 CFS ∇ TEST IOB Q = 22.1 CFS2 1.3 1.2 V 1.1 FROM DATUM (FF) V NOTE: ZERO DATUM AT 1.0 IMLT-IAL-EINSTABILIT-PR-PLANE OF MAXIMUM V ENVELOPE OF RIPRAP 13:0 .9 VA -----SURFACE (0.0) TEST 10d .8 Q=21.9CFS .7 DISTANCE V .6 V .5 .4 V .3 V .2 V D . 1 0 0.0 2 3 10 1 4 4 .8 9 -1 VELOCITY (fps) V 1 FIGURE 18






















FIGURE 30. CROSS SECTIONS FOR FINAL SCOUR AT POINT OF MAXIMUM JET CONTRACTION











NO. 359-5010

0 10 8 11 12 13 5 6 9 VELOCITY (FPS) FIGURE G. υ B 110 <u>nt</u>







ANDERSSON'S SUITARY OF VELOCITY CURVES REQUIRED DIAMETER OF STONE AS A FUNCTION OF MEAN VELOCITY

FIGURE 36



APPENDIX

COMPUTOR PROGRAM ENTITLED "RIPSHA"

10 DIM S(50), Z(50), X(50), V(50), US(50) 20 READ N 30 FOR J = 1 TO N40 READ A, B, C, W, H1, H2 50 LET V(J) = 0.670857*(H2 - H1)60 LET Z(J) = B/A70 LET X(J) = C/B80 LET $S(J) = (C *B) / A^2$ 90 NEXT J 100 LET K1 = K2 = K3 = K4 = 0110 FOR J = 1 TO N120 IF Z(J) > 0.667 THEN 200 130 IF X(J) < 0.667 THEN 170 140 LET K2 = K2 + 1 150 LET US(J) = "II" 160 GØ TØ 260 170 LET K1 = K1 + 1180 LET US(J) = "I" 190 GØ TØ 260 200 IF X(J) < 0.667 THEN 240 210 LET K4 = K4 + 1 220 LET US(J) = "IV" 230 GØ TØ 260 240 LET K3 = K3 + 1 250 LET US(J) = "III" 260 NEXT J . 270 LET S1 =S2 =S3 =S4 =S5 =S6 =S7 =S8 =S9 =T1 =0 280 FOR J = 1 TO N 290 IF S(J) > 0.1 THEN 320 $300 \text{ LET } \text{T1} = \text{T1} \div 1$ 310 GØ TØ 570 320 IF S(J) > 0.2 THEN 350 3 30 LET S8 = S8 +1 340 GØ TØ 570 350 IF S(J) > 0.3 THEN 380 360 LET S9 = S9 + 1 370 G9 TØ 570 380 IF S(J) > 0.4 THEN 410 390 LET S1 = S1 + 1400 GØ TØ 570 410 IF S(J) > 0.5 THEN 440 420 LET S2 = S2 + 1 430 GØ TØ 570 440 IF S(J) > 0.6 THEN 470 450 LET S3 = S3 + 1460 GØ TØ 570

14. 管理

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COMPUTOR PROGRAM ENTITLED "RIPSHA", continued

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470 IF S(J) > 0.7THEN 500 480 LET S4 = S4 + 1 490 GØ TØ 570 500 IF S(J) > 0.8 THEN 530 $510 \text{ LET } \text{S5} = \text{S5} \div 1$ 520 GØ TØ 570 530 IF S(J) > 0.9 THEN 560 540 LET S6 = S6 + 1 550 GØ TØ 570 560 LET S7 = S7 + 1570 NEXT J 580 LET T1 = 100 * (T1/N)590 LET S9 = 100 * (S9/N)600 LET S8 = 100 * (S8/N) 610 LET S1 = 100 * (S1/N). 620 LET S2 = 100* (S2/N) 630 LET S3 = 100* (S3/N) 640 LET S4 = 100* (S4/N) 650 LET S5 = 100* (S5/N) 660 LET S6 = 100* (S6/N) 670 LET S7 = 100* (S7/N) 680 LET J3 = 1690 PRINT 7 00 PRINT "RØCK", "ZING", "BC/At2", "VØL" 7 10 LET M1 = J3720 PRINT "NUMBER", "CLASS", "SPHERICITY", "CU FT" 7 30 LET J3 = 1740 FOR J = M1 TO N750 LET J3 = J3 + 1 760 PRINT J, US(J), S(J), V(J) 770 IF J3 > 20 THEN 690 780 NEXT J 7 90 PRINT 795 PRINT ", ", ", "TABLE" 800 PRINT

COMPUTOR PROGRAM ENTITLED "RIPSHA", continued

8 0 5	PRINT												
810	PRINT				•								
815	PRINT	11 11	, "ZII	VG CLA	SSIF	TICA	TIC	INS:	PE	R CENT	PER	CATEG	ØRY
820	PRINT	** **	, "					"	7				
825	PRINT	17	", "I'	', "II	", "	'III'	و '	"IV	117				
8 30	PRINT	71	", "D	ISK",	"SPH	ERI	CAL	",	"BLA	DE", "	RØDLI	KE"	
8 35	PRINT	PER	CENT".	. ((K1	*100	D) /N), (CKR	*100)/N).((K3*10	002/N	2:
8 40	PRINT	((K4	*100)	IN)									
8 45	PRINT				•								
850	PRINT	"SPH	ERICI	TY CLA	SSIR	TICA	TIC	IN =	· (C*	B) /A12			
855	PRINT	11											
860	PRINT	"PER	CENT	- 0.0	ΤØ	0.1	=	";	T 1				
8 65	PRINT	"PER	CENT	0.1	ΤØ	0.2	=	11;	58				
870	PRINT	"PER	CENT	- 0.2	TØ	0.3	=	11 :	59		1.1.1		
875	PRINT	"PER	CENT	- 0.3	TO	0.4	=	17 :	SI				
880	PRINT	"PER	CENT	- 0.4	TØ	0.5	=	** ;	52				
885	PRINT	"PER	CENT	0.5	ΤØ	0.6	=	17 ;	\$3				
890	PRINT	"PER	CENT	- 0.6	TØ	0.7	=	11 :	54				1.1
8 95	PRINT	PER	CENT	- 0.7	TØ	0.8	=	11 :	SS				
900	PRINT	PER	CENT	- 0.8	TØ	0.9	=	11 :	56				
9 0 5	PRINT	UPER	CENT	- 0.9	TØ	1.0	=		57	4.02			
910	PRINT								.~.				
915	GØ TØ	20								1.1.1.1			
1 000	DATA	36							4				
9999	9 END		- 4	ne.						1. 1150	21 - 2		Ser.
	8 05 8 10 8 20 8 25 8 30 8 25 8 30 8 25 8 30 8 35 8 40 8 55 8 60 8 75 8 85 8 90 9 905 9 10 9 99 9 10 9 99	8 05 PRINT 8 10 PRINT 8 10 PRINT 8 20 PRINT 8 20 PRINT 8 25 PRINT 8 30 PRINT 8 30 PRINT 8 40 PRINT 8 40 PRINT 8 40 PRINT 8 40 PRINT 8 50 PRINT 8 60 PRINT 8 60 PRINT 8 60 PRINT 8 70 PRINT 8 70 PRINT 8 70 PRINT 8 70 PRINT 8 70 PRINT 8 70 PRINT 9 70 PRINT 9 00 PRINT 9 00 PRINT 9 10 PRINT 9 10 PRINT 9 10 DATA 9 999 END	8 05 PRINT 8 10 PRINT 8 15 PRINT "" 8 20 PRINT "" 8 25 PRINT "" 8 30 PRINT " 8 30 PRINT "PER 8 40 PRINT ((K4 8 45 PRINT "PER 8 40 PRINT "SPH 8 50 PRINT "PER 8 50 PRINT "PER 8 65 PRINT "PER 8 70 PRINT "PER 8 75 PRINT "PER 8 80 PRINT "PER 8 80 PRINT "PER 8 90 PRINT "PER 8 90 PRINT "PER 8 90 PRINT "PER 9 00 PRINT "PER 9 00 PRINT "PER 9 10 PRINT "PER 9 10 PRINT 36 9 999 END	8 05 PRINT 8 10 PRINT 8 10 PRINT 8 15 PRINT " ", "ZIN 8 20 PRINT " ", "I" 8 25 PRINT " ", "I" 8 30 PRINT " ", "D 8 35 PRINT "PERCENT" 8 40 PRINT ((K4*100)) 8 45 PRINT 8 40 PRINT ((K4*100)) 8 45 PRINT 8 50 PRINT "PERCENT" 8 50 PRINT "PER CENT 8 60 PRINT "PER CENT 8 60 PRINT "PER CENT 8 70 PRINT "PER CENT 8 70 PRINT "PER CENT 8 80 PRINT "PER CENT 8 80 PRINT "PER CENT 8 90 PRINT "PER CENT 8 90 PRINT "PER CENT 9 00 PRINT "PER CENT 9 00 PRINT "PER CENT 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	8 05 PRINT 8 10 PRINT 6 15 PRINT " ", "ZING CLA 8 20 PRINT " ", "I", "II 8 25 PRINT " ", "I", "II 8 30 PRINT " ", "DISK", 8 35 PRINT "PERCENT", ((K1 8 40 PRINT ((K4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLA 8 50 PRINT "PER CENT - 0.0 8 60 PRINT "PER CENT - 0.0 8 65 PRINT "PER CENT - 0.1 8 70 PRINT "PER CENT - 0.2 8 75 PRINT "PER CENT - 0.3 8 80 PRINT "PER CENT - 0.4 8 85 PRINT "PER CENT - 0.5 8 90 PRINT "PER CENT - 0.5 9 00 PRINT "PER CENT - 0.7 9 00 PRINT "PER CENT - 0.9 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIF 8 20 PRINT " ", "I", "II", " 8 25 PRINT " ", "I", "II", " 8 30 PRINT " ", "DISK", "SPF 8 35 PRINT "PERCENT", ((K1*100 8 40 PRINT ((K4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIF 8 50 PRINT "PER CENT - 0.0 TØ 8 60 PRINT "PER CENT - 0.1 TØ 8 60 PRINT "PER CENT - 0.2 TØ 8 70 PRINT "PER CENT - 0.3 TØ 8 70 PRINT "PER CENT - 0.3 TØ 8 80 PRINT "PER CENT - 0.4 TØ 8 85 PRINT "PER CENT - 0.5 TØ 8 90 PRINT "PER CENT - 0.6 TØ 9 90 PRINT "PER CENT - 0.7 TØ 9 00 PRINT "PER CENT - 0.9 TØ 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIFICA 8 20 PRINT " ", "I", "II", "III" 8 25 PRINT " ", "I", "II", "III" 8 30 PRINT " ", "DISK", "SPHERIC 8 35 PRINT "PERCENT", ((K1*100)/N) 8 40 PRINT ((K4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICA 8 50 PRINT "PER CENT - 0.0 TØ 0.1 8 60 PRINT "PER CENT - 0.1 TØ 0.2 8 70 PRINT "PER CENT - 0.2 TØ 0.3 8 75 PRINT "PER CENT - 0.3 TØ 0.4 8 80 PRINT "PER CENT - 0.4 TØ 0.5 8 85 PRINT "PER CENT - 0.5 TØ 0.6 8 90 PRINT "PER CENT - 0.6 TØ 0.7 8 95 PRINT "PER CENT - 0.8 TØ 0.9 9 05 PRINT "PER CENT - 0.9 TØ 1.0 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIFICATION 8 20 PRINT " ", "I", "II", "III", 8 25 PRINT " ", "DISK", "SPHERICAL 8 35 PRINT "PERCENT", ((K1*100)/N), (8 40 PRINT ((K4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICATION 8 45 PRINT 8 50 PRINT "PER CENT - 0.0 TØ 0.1 = 8 60 PRINT "PER CENT - 0.1 TØ 0.2 = 8 70 PRINT "PER CENT - 0.2 TØ 0.3 = 8 75 PRINT "PER CENT - 0.3 TØ 0.4 = 8 80 PRINT "PER CENT - 0.4 TØ 0.5 = 8 85 PRINT "PER CENT - 0.5 TØ 0.6 = 8 90 PRINT "PER CENT - 0.7 TØ 0.8 = 9 00 PRINT "PER CENT - 0.8 TØ 0.9 = 9 05 PRINT "PER CENT - 0.9 TØ 1.0 = 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIFICATIØNS: 8 20 PRINT " ", "I", "II", "III", "IV 8 30 PRINT " ", "DISK", "SPHERICAL", 8 35 PRINT "PERCENT", ((K1*100)/N),((K2 8 40 PRINT ((K4*100)/N)) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICATIØN = 8 55 PRINT "PER CENT - 0.0 TØ 0.1 = "; 8 60 PRINT "PER CENT - 0.1 TØ 0.2 = "; 8 70 PRINT "PER CENT - 0.2 TØ 0.3 = "; 8 75 PRINT "PER CENT - 0.3 TØ 0.4 = "; 8 80 PRINT "PER CENT - 0.5 TØ 0.6 = "; 8 80 PRINT "PER CENT - 0.6 TØ 0.7 = "; 8 90 PRINT "PER CENT - 0.7 TØ 0.8 = "; 9 00 PRINT "PER CENT - 0.8 TØ 0.9 = "; 9 05 PRINT "PER CENT - 0.9 TØ 1.0 = "; 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIFICATIØNS: PE 8 20 PRINT " ", "I", "II", "III", "IV" 8 30 PRINT " ", "DISK", "SPHERICAL", "BLA 8 35 PRINT "PERCENT", ((K1*100)/N),((K2*100) 8 40 PRINT ((K4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICATIØN = (C* 8 55 PRINT "SPHERICITY CLASSIFICATIØN = (C* 8 60 PRINT "PER CENT - 0.0 TØ 0.1 = "; T1 8 65 PRINT "PER CENT - 0.1 TØ 0.2 = "; S8 8 70 PRINT "PER CENT - 0.2 TØ 0.3 = "; S9 8 75 PRINT "PER CENT - 0.3 TØ 0.4 = "; S1 8 80 PRINT "PER CENT - 0.4 TØ 0.5 = "; S2 8 85 PRINT "PER CENT - 0.5 TØ 0.6 = "; S3 8 90 PRINT "PER CENT - 0.7 TØ 0.8 = "; S5 9 00 PRINT "PER CENT - 0.8 TØ 0.9 = "; S6 9 05 PRINT "PER CENT - 0.9 TØ 1.0 = "; S7 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END	<pre>8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIFICATIØNS: PER CENT 8 20 PRINT " ", "I", "II", "III", "IV" 8 25 PRINT " ", "DISK", "SPHERICAL", "BLADE", " 8 35 PRINT "PERCENT", ((K1*100)/N),((K2*100)/N),(8 40 PRINT ((K4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICATIØN = (C*B)/A*2" 8 50 PRINT "SPHERICITY CLASSIFICATIØN = (C*B)/A*2" 8 60 PRINT "PER CENT - 0.0 TØ 0.1 = "; T1 8 65 PRINT "PER CENT - 0.1 TØ 0.2 = "; S8 8 70 PRINT "PER CENT - 0.2 TØ 0.3 = "; S9 8 75 PRINT "PER CENT - 0.3 TØ 0.4 = "; S1 8 80 PRINT "PER CENT - 0.4 TØ 0.5 = "; S2 8 85 PRINT "PER CENT - 0.6 TØ 0.7 = "; S4 8 95 PRINT "PER CENT - 0.6 TØ 0.7 = "; S4 8 95 PRINT "PER CENT - 0.8 TØ 0.9 = "; S6 9 00 PRINT "PER CENT - 0.9 TØ 1.0 = "; S7 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END</pre>	<pre>8 05 PRINT 8 10 PRINT 6 15 PRINT " ", "ZING CLASSIFICATIØNS: PER CENT PER 0 8 20 PRINT " ", "I", "II", "III", "IV" 8 30 PRINT " ", "DISK", "SPHERICAL", "BLADE", "RØDLI] 8 35 PRINT " PERCENT", (CK1*100)/N),(CK2*100)/N),(CK3*10 8 40 PRINT (CK4*100)/N) 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICATIØN = (C*B)/At2" 8 50 PRINT "PER CENT - 0.0 TØ 0.1 = "; T1 8 65 PRINT "PER CENT - 0.1 TØ 0.2 = "; S8 8 70 PRINT "PER CENT - 0.2 TØ 0.3 = "; S9 8 75 PRINT "PER CENT - 0.3 TØ 0.4 = "; S1 8 80 PRINT "PER CENT - 0.4 TØ 0.5 = "; S2 8 85 PRINT "PER CENT - 0.5 TØ 0.6 = "; S3 8 90 PRINT "PER CENT - 0.6 TØ 0.7 = "; S4 8 95 PRINT "PER CENT - 0.8 TØ 0.9 = "; S6 9 05 PRINT "PER CENT - 0.9 TØ 1.0 = "; S7 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END</pre>	<pre>8 05 PRINT 8 10 PRINT 8 15 PRINT " ", "ZING CLASSIFICATIØNS: PER CENT PER CATEG 8 20 PRINT " ", "I", "II", "III", "IV" 8 25 PRINT " ", "DISK", "SPHERICAL", "BLADE", "RØDLIKE" 8 30 PRINT " ", "DISK", "SPHERICAL", "BLADE", "RØDLIKE" 8 40 PRINT "PERCENT", (CK1*100)/N),(CK2*100)/N),(CK3*100)/N 8 45 PRINT 8 50 PRINT "SPHERICITY CLASSIFICATIØN = (C*E)/A†2" 8 55 PRINT "PER CENT - 0.0 TØ 0.1 = "; T1 8 60 PRINT "PER CENT - 0.1 TØ 0.2 = "; S8 8 70 PRINT "PER CENT - 0.2 TØ 0.3 = "; S9 8 75 PRINT "PER CENT - 0.4 TØ 0.5 = "; S1 8 80 PRINT "PER CENT - 0.4 TØ 0.5 = "; S2 8 85 PRINT "PER CENT - 0.5 TØ 0.6 = "; S3 8 90 PRINT "PER CENT - 0.6 TØ 0.7 = "; S4 8 95 PRINT "PER CENT - 0.8 TØ 0.9 = "; S6 9 00 PRINT "PER CENT - 0.9 TØ 1.0 = "; S7 9 10 PRINT 9 15 GØ TØ 20 1 000 DATA 36 9 999 END</pre>

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COMPUTOR PROGRAM ENTITLED "RIPRAP"

10 DIM P(50), F(50), Ø(50), D(50), V(50) 20 DIM W(50), G(50) 30 DIM E(40), A(40), B(40), C(40), M(40) 40 READ N.Y 50 LET G1 = 060 FOR J = 1 TO N70 READ A, B, C, W, H1, H2 74 IF J > 16 THEN 80 76 LET W = W - 0.1180 LET V(J) = 0.670857*(H2-H1) 90 LET G(J) = W/V(J)100 LET G1 = G1 + G(J)110 LET W(J) = W112 LET A(J) = A114 LET B(J) = B116 LET C(J) = C120 NEXT J 130 LET G = G1/N140 FØR J = 1 TØ N150 LET D(J) = (W(J)/(.76*G))*.3333 160 LET D(J)= 12* D(J) 170 LET E(J) = (A(J)+B(J)+C(J))/3180 LET P(J) = E(J)/D(J) . 190 LET F(J) = E(J)/Y200 LET M(J) = (A(J)*B(J)*C(J))/12*3 210 LET Ø(J) =V(J)/M(J) 220 NEXT J 230 LET D1 = 0240 FOR J = 1 TO N250 LET D1 = D1 + D(J)260 NEXT J 270 LET J2 = 1280 PRINT 290 PRINT " ", " ", "TABLE" 300 PRINT

COMPUTOR PROGRAM ENTITLED "RIPRAP", continued

310 PRINT " ", "PIE TERMS FØR PHYSICAL CHARACTERISTICS ØF RIPRAP" 320 LET M1 = J2330 PRINT 340 LET J2 = 1350 PRINT "WHERE DIA=DIAMETER OF SPHERE OF WEIGHT EQUAL TO ROCK SAMPLE" 360 PRINT "PIE 1 IS RATIO OF MEAN AXIS DIM AND DIA OF = WT SPHERE" 370 PRINT "PIE 2 IS RATIØ ØF MEAN AXIS DIM AND SIGNIFICANT DEPTH" 380 PRINT "PIE 3 IS RATIØ ØF RØCK VØLUME AND ABC RECTANGULAR SØLID" 390 PRINT 400 PRINT "ROCK", "DIA", "PIE 1", "PIE 2", "PIE 3" 410 PRINT "NUMBER", "INCHES" '420 FØR J = M1 TØ N $430 \text{ LET } J2 = J2 \div 1$ 440 PRINT J.D(J),P(J),F(J),Ø(J) 450 IF J2 > 20 THEN 280 460 NEXT J 470 PRINT "MEAN DIA ØF = WEIGHT SPHERE"; (D1/N);" INCHES" 480 PRINT 490 GØ TØ 40 998 REM INSERT DATA IN LINE 1000: N, Y, G 1005 REM . AXIAL DIMENSIONS (IN) WEIGHT HT OF EQ CYL 1007 REM A C LBS B H1 H2 9999 END

N

COMPUTOR PROGRAM ENTITLED "AREVEL"

10	DIM A(20), D(20), G(20), 0(20), V(20), T(20), F(20), E(20), U(20), H(20)
15	DIM L(10), P(10), R(10), S(10), X(10), Y(10), C(10)
20	READ N
22	FOR J = 1 TO N
24	READ D(J)
281	NEXT J
30	FØR J=1 TØ N
40	READ $T(J) = O(J) = P(J) = R(J) = S(J) = X(J) = Y(J) = C(J) = O(J)$
501	LET $A(J) = P(J) - O(J)$
601	LET L(J) = A(J)/2
701	LET $A(J) = .5 * A(J)$
301	LET $V(J) = ((S(J) - R(J))*0.645)/D(J)$
901	LET $F(J) = V(J) * A(J)$
1 00	LET $O(J) = .5 * O(J)$
110	LET G(J) = O(J) - F(J)
1 30	LET $H(J) = X(J) - Y(J)$
1 40	LET $U(J) = (1/SOR(1 - (Y(J)/X(J))) + SOR(2*32.2*H(J))$
1.50	LET $E(J) = U(J) * C(J) * A(J)$.
1 60	NEXT J
170	PRINT
180	PRINT " ", " ", "TABLE"
1 90	PRINT
2.00	PRINT " ", "DISCHARGE"
210	PRINT
220	PRINT "WHERE", "G = FLOW IN CUBIC FEET PER SECOND IN GRAVEL"
230	PRINT " ", "0 = TOTAL DISCHARGE FOR UNIT WIDTH SECTION"
2 40	PRINT " ", "F = DISCHARGE IN JET COMPUTED BY THE CONTINUITY EQUATION"
250	PRINT " ", "E = DISCHARGE IN JET COMPUTED BY EQUATION 3.16"
2 60	PRINT
270	PRINT "TEST", "DISCHARGE", "TOTAL", "DISCHARGE", "DISCHARGE"
2.80	PRINT "NUMBER", "IN", "DISCHARGE", "F", "E"
2.90	PRINT ", "RIPRAP", "IN", " ", "
3 00	PRINT ", "G", "FLUME", " ", "

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COMPUTOR PROGRAM ENTITLED "AREVEL". (continued)

```
310 PRINT " ">"(CFS/FT)">"(CFS/FT)">"(CFS/FT)","(CFS/FT)"
320 PRINT
330 FGR J = 1 TO N
3 40 PRINT T(J),G(J),G(J), F(J), E(J)
350 NEXT J
360 PRINT
370 PRINT
3 20 PRINT " "," "," TABLE"
390 PRINT
400 PRINT " ", "COMPARISON OF VELOCITY COMPUTATION"
410 PRINT
420 PRINT"WHERE V = MEAN VELOCITY OF JET BY VELOCITY PROFILE AVERAGING"
430 PRINT TAB(7);"U = MEAN JET VELOCITY BY EQUATION 3.15"
435 PRINT " " ,"L = MEAN DEPTH OF JET ABOVE RIPRAP SURFACE"
440 PRINT
450 PRINT "TEST", "V", "U", "L"
460 PRINT "NUMBER", "(FPS)", "(FPS)"
470 PRINT
480 FOR J= 1 TO N
490 PRINT T(J), V(J), U(J), L(J)
500 NEXT J
510 GØ TØ 20
520 DATA 6
525 DATA 1.75, 1.96, 1.35, 1.35, 1.42, 1.66
530 DATA 10, 81.70, 83.78, 77.30, 95.52, 4.10, 3.090, .81000, 21.85
540 DATA 11, 99.78, 99.90, 06.04, 23.77, 3.85, 3.314, .89000, 22.09
550 DATA 14, 12.14, 14.82, 80.02, 90.34, 4.20, 2.057, .83330, 22.30
560 DATA 15, 06.51, 08.72, 90.34, 108.12, 3.70, 2.112, .65120, 22.00
570 DATA 19, 94.05, 97.80, 14.85, 31.00, 3.80, 2.619, .83188, 21.47
580 DATA 20, 88.76, 90.07, 54.55, 73.95, 5.14, 2.683, .67860, 21.82
1000 END
```

N

CALIFORNIA SPECIFICACIONS





BANK AND SHORE PROTECTION

	Shape	ratios			$k = \frac{t}{\sqrt[3]{w}} \text{for}$		
Shape	r	8	factor V/Vm	$j = \frac{V}{l^4}$	sg = 2.4 d = 150	2.8 175	
Spheroidal	1.00	1.00	.52		:23		
Lenticular	.33	1.00	.52	.17	.34	.32	
Ilipsoidal.	.33	.33	.52	.058	.49	.46	
'ylindrical	.33	,33	.785	.087	.42	.40	
Discoidal	.33	1.00	.785	.26	.30	.28	
uboidal	1.00	1.00	.19	. 19	.33	.31	
'etragonal	.33	.75	.26	.082	.43	.41	
labal	.33	.97	.28	.092	,41	,30	
Conoidal	1.00	1.00	.26	.26	.30	.28	
yramidal.	1.00	1.00	.33	.33	,27	.26	
'etrahedral	.33	.38	.17	.062	.68	.63	
Normal cube	1.00	1.00	1.00	1.00	.188	.179	

TABLE 4. EFFECT OF EXTREME RANGE OF SHAPE ON SIZE OF SPECIFIED REVETMENT STONES

For this purpose, the characteristic shapes are assumed to be the extremes within the definitions of shapes and specifications for permissible elongation. Such shapes may occur, but they will not be dominant, except possibly the slabal shape in rock broken from laminar formations, and the lenticular shape common to cobbles.

Axial length is also taken as an extreme. The effect of this is most striking for the cuboidal shape. If axes were taken normal to faces of a cube, the volume is l^3 , but if l is the diagonal of the cube, its volume is $.19l^3$. The normal cube is listed at bottom of the table to emphasize this difference.

Maximum Length

Table 4 expresses the volume of extreme shapes in two ways: (1) as a shape factor, or ratio to the enveloping prism, and (2) as a ratio to the cube of the longest dimension.

For the latter, the table shows k for the relation $1 = j l^3$, and since W = Vd,

$$l = \sqrt[3]{W/jd} = k \sqrt[3]{W}$$
 where $k = 1/\sqrt[3]{jd}$

Values of k for extremes of density are tabulated in the last two columns.

To illustrate use of k, consider a 4-ton stone, for which the cube root of the weight in pounds is 20. Using a density of 175 pcf, if the stone is a perfect cube, its side is $20 \ge 1.79 = 3.58$ ft. If a sphere, its diameter is $20 \ge 2.2 = 4.4$ ft. If lenticular with vertical diameter only a third of its horizontal diameter, the latter is $20 \ge 3.2 = 6.4$ ft. Specifications will even allow a shape $20 \ge .46 = 9.2$ ft long.

